

# Performance evaluation for inerter-based dynamic vibration absorbers

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## Abstract

This paper is concerned with the  $H_\infty$  and  $H_2$  optimization problem for inerter-based dynamic vibration absorbers (IDVAs). The proposed IDVAs are obtained by replacing the damper in the traditional dynamic vibration absorber (TDVA) with some inerter-based mechanical networks. It is demonstrated in this paper that adding one inerter alone to the TDVA provides no benefits for the  $H_\infty$  performance and negligible improvement (less than 0.32% improvement over the TDVA when the mass ratio less than 1) for the  $H_2$  performance. This implies the necessity of introducing another degree of freedom (element) together with inerter to the TDVA. Therefore, four different IDVAs are proposed by adding an inerter together with a spring to the TDVA, and significant improvement for both the  $H_\infty$  and  $H_2$  performances is obtained. Numerical simulations in dimensionless form show that more than 20% and 10% improvement can be obtained for the  $H_\infty$  and  $H_2$  performances, respectively. Besides, for the  $H_\infty$  performance, the effective frequency band can be further widened by using inerter.

*Keywords:* Inerter, dynamic vibration absorber,  $H_\infty$  optimization,  $H_2$  optimization.

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## 1. Introduction

1 Dynamic vibration absorber (DVA) is an auxiliary mass system attached to a vibrating  
2 primary system to reduce undesired vibration, which is widely used in the fields of civil  
3 and mechanical engineering for its simple design and high reliability [1]. In the first DVA  
4 proposed by Frahm in 1909 [2], only a spring was employed, and it was useful only in a  
5 narrow band of frequency. In 1928, the damping mechanism was introduced by Ormondroyd  
6 and Den Hartog [3], which is a parallel arrangement of a spring and a damper, and as a  
7 result, the effective frequency band was significantly widened. It was also pointed out in [3]  
8 that for the spring-damper DVA (in this paper, it is called the traditional DVA or TDVA)  
9 and undamped primary system, there were two frequencies called fixed points, where the  
10 magnitudes were independent of the damping, and the optimal setting of the spring stiffness  
11 was the one equalizing the magnitudes at the fixed points, and the optimal damping was the  
12 one making the curves of the frequency response horizontally pass through the fixed points.  
13 Such a tuning method is still in use today and currently known as the fixed-point method [1],  
14 which has been demonstrated to be a suboptimal  $H_\infty$  optimization method [4]. The exact

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15 solutions were analytically derived in [4] and it was also shown that the fixed-point method  
16 actually yielded an approximate but highly precise solution (with less than 0.5% deviation  
17 when the mass ratio less than 1). Another common performance measure of tuning DVA is the  
18  $H_2$  performance measure, which is desirable when the primary system subjected to random  
19 excitations. The objective of  $H_2$  optimization is to optimize the total vibration energy of  
20 the system over all frequencies [5]. For the TDVA with undamped primary systems, the  
21 optimal tuning frequency and damping ratio were investigated in [5], and then the analytical  
22 solutions were derived in [6]. For damped primary systems, various design methods and  
23 tuning criteria have been proposed, such as those in [7, 9, 8, 10], and the applications of the  
24 TDVA in nonlinear and distributed primary systems have been investigated [11, 12, 13]. The  
25 active DVAs utilizing feedback control actions have also been proposed [14, 15, 16].

26 Inerter is a two-terminal mechanical device proposed by Smith in 2002 [17], which has the  
27 property that the applied force at its two terminals is proportional to the relative acceleration  
28 between them and the constant of proportionality is called inertance with a unit of kilogram.  
29 Inerter has been successfully applied in Formula One racing car suspension systems [18],  
30 and now, applications of inerter in various mechanical systems, such as vehicle suspensions  
31 [19, 20, 21, 22] and vibration suppression systems [23, 24, 25, 26], have been investigated.  
32 Recently, the reduction of vibration systems' natural frequencies by using inerter has been  
33 theoretically demonstrated [27], and the interest in passive network synthesis has also been  
34 rekindled [28, 29, 30, 31, 32, 33, 34, 35, 38].

35 Vibration absorption is one of the potential applications of inerter [17]. In [17], the  
36 problem of designing inerter-based networks to absorb vibration at a specific frequency was  
37 studied. Thereafter, the suppression of vibration over a broadband frequency by using inerter  
38 has been proposed. In [23], an inerter-based configuration ( $C4$  in this paper) was employed  
39 between adjacent storeys to suppress the vibration of a multi-storey building. In [24], optimal  
40 solutions for several inerter-based isolators (including all the configurations except  $C5$  in this  
41 paper) were algebraically derived based on a "uni-axial" vibration isolation system. In [25],  
42 a new configuration incorporating an inerter was proposed and applied to a mechanical  
43 cascaded (chain-like) systems. In [26], the dynamics of a tuned mass absorber with an  
44 additional viscous damper and an inerter attached to the pendulum was investigated.

45 In this paper, a novel structure for inerter-based DVAs (IDVAs) is proposed by replacing  
46 the damper in the TDVA with some inerter-based mechanical networks, and both the  $H_\infty$   
47 and  $H_2$  performances of the proposed IDVAs are investigated. It is demonstrated in this  
48 paper that adding an inerter alone to the TDVA, no matter it is in parallel connection or  
49 in series connection, provides no benefits for the  $H_\infty$  performance and negligible benefits  
50 (less than 0.32% improvement over the TDVA when the mass ratio less than 1) for the  $H_2$   
51 performance. In contrast, by adding an inerter together with a spring to the TDVA (e.g.  $C3$ ,  
52  $C4$ ,  $C5$ , and  $C6$  in this paper), both  $H_\infty$  and  $H_2$  performances can be significantly improved.  
53 Over 20% improvement compared with the TDVA can be obtained for the  $H_\infty$  performance,  
54 and the effective frequency band can also be further widened by using inerter. For the  $H_2$   
55 performance, it is analytically demonstrated that the IDVAs proposed in this paper perform  
56 surely better than the TDVA and over 10% improvement is obtained in numerical simulation.  
57 Moreover, a minmax framework directly using the resonance frequencies is proposed for the  
58  $H_\infty$  optimization, and an algebraic method to analytically calculate the  $H_2$  norm is employed  
59 for the  $H_2$  optimization. All these constitute the main contributions of this paper.

60 The organization of this paper is as follows. In Section 2, the IDVAs in this paper  
 61 are introduced and the dimensionless representations of displacement transfer functions are  
 62 derived. In Section 3 and Section 4, the  $H_\infty$  and  $H_2$  optimization problems are solved for the  
 63 IDVAs and the comparison between the IDVAs and the TDVA is conducted. Conclusions  
 64 are drawn in Section 5.

## 65 2. Inerter-based dynamic vibration absorbers

66 Fig. 1 shows the comparison between the IDVAs proposed in this paper and the TDVA,  
 67 where the IDVA is obtained by replacing the damper in the TDVA with some inerter-based  
 68 mechanical networks. The entire networks employed in this paper are shown in Fig. 2. The  
 69 equations of motion for the whole system in the Laplace domain are

$$Ms^2x = F + F_d - Kx, \quad (1)$$

$$ms^2x_a = -F_d, \quad (2)$$

$$F_d = (k + sY(s))(x_a - x), \quad (3)$$

70 where  $Y(s)$  is the admittance of the inerter-based passive mechanical networks and  $F_d$  is the  
 71 force of the DVA imposed on the primary mass  $M$ .

72 From (2) and (3), one obtains,

$$F_d = -R(s)x,$$

73 where

$$R(s) = \frac{(k + sY(s))ms^2}{k + ms^2 + sY(s)}.$$

74 Then, one obtains the displacement transfer function as

$$H(s) = \frac{x}{x_s} = \frac{1}{\frac{s^2}{\omega_n^2} + \frac{1}{K}R(s) + 1}, \quad (4)$$

75 where  $x_s = F/K$  and  $\omega_n = \sqrt{\frac{K}{M}}$  are the static displacement and natural frequency of the  
 76 primary system, respectively.

77 The admittance of each network in Fig. 2 is shown in Table 1, where  $Y_i(s)$ ,  $i = 1, \dots, 6$   
 78 corresponds to  $C_i$ ,  $i = 1, \dots, 6$  in Fig. 2, respectively. Substituting each  $Y_i(s)$  into (4), one  
 79 can obtain the detailed transfer function for each configuration. To obtain the dimensionless  
 80 representation of each configuration, the following dimensionless parameters are defined as

$$\left. \begin{aligned} \mu &= \frac{m}{M} : \text{mass ratio} \\ \delta &= \frac{b}{m} : \text{inertance-to-mass ratio} \\ \zeta &= \frac{c}{2\sqrt{mk}} : \text{damping ratio} \\ \eta &= \frac{\omega_b}{\omega_m} : \text{corner frequency ratio} \\ \gamma &= \frac{\omega_m}{\omega_n} : \text{natural frequency ratio} \\ \lambda &= \frac{\omega}{\omega_n} : \text{forced frequency ratio} \end{aligned} \right\} \quad (5)$$

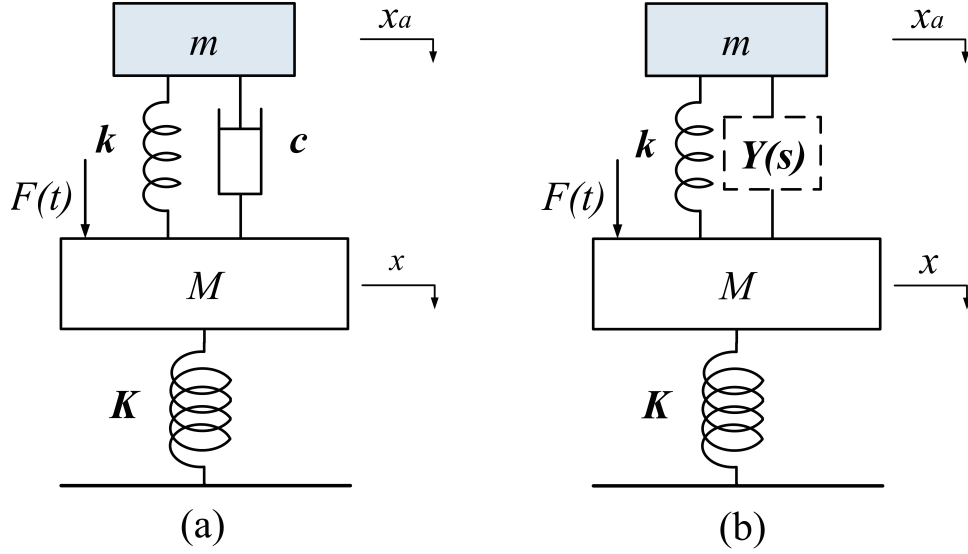


Figure 1: Dynamic vibration absorbers (DVA): (a) traditional dynamic vibration absorber (TDVA); (b) inerter-based dynamic vibration absorber (IDVA).

81 where

$$\left. \begin{aligned}
 \omega_m &= \sqrt{\frac{k}{m}} : \text{natural frequency of the DVA} \\
 \omega_b &= \sqrt{\frac{k_1}{b}} : \text{corner frequency of the DVA} \\
 \omega_n &= \sqrt{\frac{K}{M}} : \text{natural frequency of the primary system}
 \end{aligned} \right\} \quad (6)$$

82 **Remark 1.** *In this paper, the force-current analogy between mechanical and electrical net-*  
 83 *works is employed, and admittance is defined to be the ratio of force to velocity, which agrees*  
 84 *with the usual electrical terminology (see [17] for details). Such a definition is consistent with*  
 85 *some books [36, p. 328], but not others which use the force-voltage analogy [37].*

86 **Remark 2.** *Since the natural frequencies would be perturbed by using inerter as demonstrated*  
 87 *in [27],  $\omega_m$  and  $\omega_n$  are not the real natural frequencies of the whole system. Neither is*  
 88  *$\omega_b$  the real corner frequency. Here, these notations are employed just for dimensionless*  
 89 *representations.*

90 Replacing  $s$  with  $j\omega$  in (4), the frequency response functions in a dimensionless form can  
 91 be obtained as

$$H_i(j\lambda) = \frac{R_{ni} + jI_{ni}}{R_{mi} + jI_{mi}}, \quad i = 1, \dots, 6, \quad (7)$$

92 where  $R_{ni}$ ,  $I_{ni}$ ,  $R_{mi}$ , and  $I_{mi}$ ,  $i = 1, \dots, 6$  are functions with respect to  $\lambda$ ,  $\gamma$ ,  $\delta$ , and  $\zeta$ . The  
 93 detailed representations are given in Appendix A.

### 94 3. $H_\infty$ optimization for the IDVAs

#### 95 3.1. Minmax optimization problem formulation

96 The objective of the  $H_\infty$  optimization is to minimize the maximum magnitude of the  
 97 frequency response  $|H_i(j\lambda)|$ ,  $i = 1, \dots, 6$ , which is known as the  $H_\infty$  norm of  $H_i(s)$  with

Table 1: Admittance  $Y(s)$  for each configuration in Fig. 2.

$Y_1(s) = bs + c$	$Y_2(s) = \frac{1}{\frac{1}{bs} + \frac{1}{c}}$	$Y_3(s) = \frac{1}{\frac{s}{k_1} + \frac{1}{c} + \frac{1}{bs}}$
$Y_4(s) = \frac{1}{\frac{1}{\frac{k_1}{s} + c} + \frac{1}{bs}}$	$Y_5(s) = \frac{1}{\frac{1}{\frac{k_1}{s} + bs} + \frac{1}{c}}$	$Y_6(s) = \frac{1}{\frac{1}{bs+c} + \frac{s}{k_1}}$

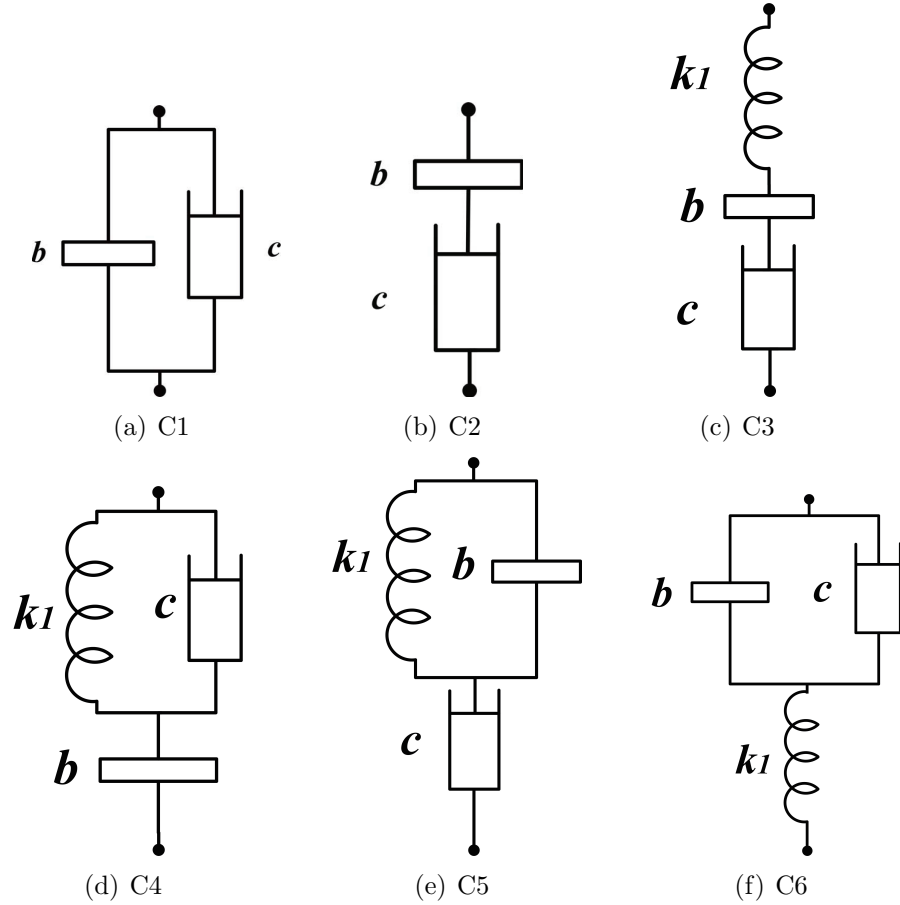


Figure 2: The employed inverter-based networks as  $Y(s)$  in Fig. 1.

98  $s = j\lambda$ . For the TDVA, the fixed-point method [1] is commonly used to analytically obtain  
 99 the optimal parameters [1, Chapter 3.3]. Since there always exist more than two fixed points  
 100 with respect to the damping ratio for IDVAs, it is difficult to obtain simple and analytical  
 101 representations for optimal parameters. Given this fact, in this paper, a minmax optimization  
 102 problem is formulated as follows to directly minimize the magnitude at resonance frequencies.

103 For a given mass ratio  $\mu$ , solving the follow minmax problem

$$\min_{\delta, \gamma, \eta, \zeta} \left( \max_{\lambda_l} (|H_i(j\lambda_l)|) \right), i = 1, \dots, 6 \quad (8)$$

104 subject to  $\delta \geq 0$ ,  $\gamma \geq 0$ ,  $\eta \geq 0$ ,  $\zeta \geq 0$ , and  $\lambda_l$ ,  $l = 1, \dots, N$ , are the real and positive solutions  
 105 of the following equation

$$\frac{\partial |H_i(j\lambda)|^2}{\partial \lambda^2} = 0, \quad (9)$$

106 where  $i = 1, \dots, 6$  corresponds to the six IDVAs in Fig. 2, respectively.

107 The underlying idea of the minmax problem (8) and (9) is, instead of using the fixed  
 108 points to approximately minimize the  $H_\infty$  norm as done in the fixed-point method [1], here  
 109 the resonance frequencies are directly used to exactly minimize the  $H_\infty$  norm. This is inspired  
 110 by the method in [4], where the two resonance frequencies were employed to derive the  
 111 exact solutions for the TDVA. Note that the solution set of (9), that is  $\lambda_l$ ,  $l = 1, \dots, N$ ,  
 112 contains the resonance frequencies, anti-resonance frequencies, and other frequencies where  
 113 the curves horizontally pass through. Since the largest magnitude of the frequency response,  
 114 representing the  $H_\infty$  norm of the transfer function, only occurs at resonance frequencies, it  
 115 is sufficient to minimize  $\max_{\lambda_l} (|H_i(j\lambda_l)|)$ ,  $l = 1, \dots, N$ , to obtain the optimal  $H_\infty$  norm of  
 116 the transfer function  $H_i(s)$ .

117 Equation (9) can be transformed into a polynomial function with respect to  $\lambda^2$  as follows.  
 118 From (7),  $|H_i(j\lambda)|^2$  can be written as

$$|H_i(j\lambda)|^2 = \frac{n}{m},$$

119 where  $n = R_{ni}^2 + I_{ni}^2$ ,  $m = R_{mi}^2 + I_{mi}^2$ . Since

$$\frac{\partial |H_i(j\lambda)|^2}{\partial \lambda^2} = \frac{n'm - m'n}{m^2},$$

120 where  $n' = \frac{\partial n}{\partial \lambda^2}$  and  $m' = \frac{\partial m}{\partial \lambda^2}$ , (9) is equivalent to

$$n'm - m'n = 0, \quad (10)$$

121 which is an equation of  $\lambda^2$  with different orders for different configurations.

122 Problem (8) and (10) is a constrained optimization problem, and the equality constraint  
 123 (10) can be transformed into the objective function by employing  $\lambda_l = f(\delta, \gamma, \eta, \zeta)$ . In this  
 124 paper, a direct search method is employed to solve the constrained optimization problem (8)  
 125 and (10) by using the Matlab solver *patternsearch* with multiple starting points.

126 *3.2. Comparison between the TDVA and IDVAs*

127 For the TDVA, the optimal parameters can be analytically obtained as [1]:

$$\gamma_{opt} = \sqrt{\frac{1}{1+\mu}}, \quad \zeta_{opt} = \sqrt{\frac{3\mu}{8(1+\mu)}},$$

128 and the optimal height at the two fixed points are  $\sqrt{\frac{2+\mu}{\mu}}$ .

129 *3.2.1. Performance limitation of C1 and C2*

130 In this subsection, it will be demonstrated that configurations *C1* and *C2* provide no  
131 improvement for the  $H_\infty$  performance compared with the TDVA.

132 For configuration *C1*, by directly using the fixed-point method in [1], the optimal param-  
133 eters for *C1* can be analytically obtained as

$$\gamma_{opt} = \frac{\sqrt{1+(1+\mu)\delta}}{1+\mu}, \quad \zeta_{opt} = \sqrt{\frac{3\mu}{8(1+\mu)}},$$

134 and the optimal height at the two fixed points are  $\sqrt{\frac{2+\mu+2\delta(1+\mu)}{\mu}}$ . It is obvious that the optimal  
135  $\delta$  is 0, which means that the parallel inerter in configuration *C1* provides no improvement in  
the  $H_\infty$  optimization. Such an observation is shown in Fig. 3 with  $\mu = 0.1$ .

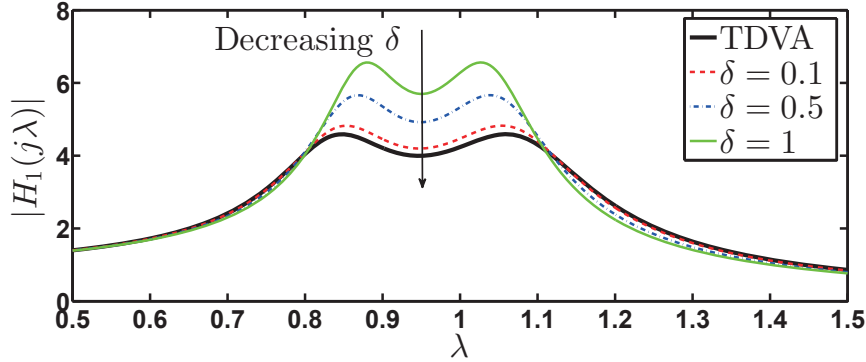


Figure 3: Comparison between the TDVA and *C1* when  $\mu = 0.1$  with different  $\delta$ .

136

137 The minmax optimization method proposed in this paper is also applicable for *C1* and a  
138 comparison between the method in this paper and the fixed-point method is shown in Fig. 4.  
139 As shown in Fig. 4, the results by these two methods highly coincide with each other and  
140 the results are consistent with the analytical solutions in [4, Table 2], which demonstrates  
141 the effectiveness of the method in this paper.

142 In what follows, it will be shown that for configuration *C2*, the series-connected inerter  
143 provides no improvement for the  $H_\infty$  performance as well. To show the influence of  $\delta$ , the  
144 problem (8) is slightly modified as: for a given  $\mu$  and  $\delta$ ,

$$\min_{\gamma, \zeta} \left( \max_{\lambda_l} (|H_2(j\lambda_l)|) \right),$$

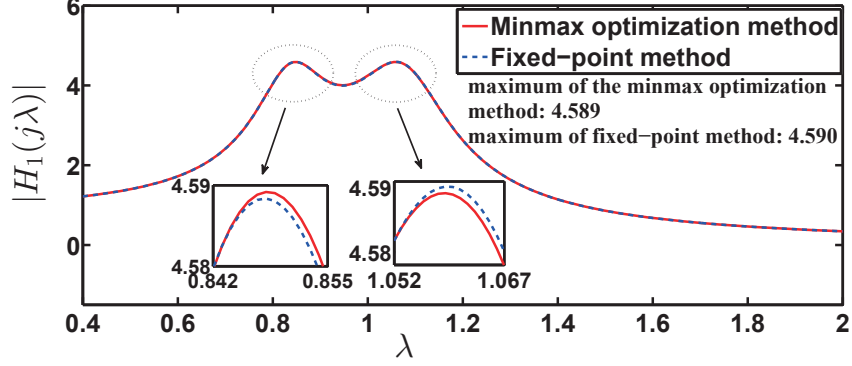


Figure 4: Comparison between the minmax optimization method in this paper and the fixed-point method when  $\mu = 0.1$ .

145 subject to  $\gamma \geq 0$ ,  $\eta \geq 0$ ,  $\zeta \geq 0$ , and  $\lambda_l$ ,  $l = 1, \dots, N$ , are the real and positive solutions of  
 146 (10). Fig. 5 shows the comparison between  $C2$  with different  $\delta$  and the TDVA when  $\mu = 0.1$ ,  
 147 where it is clearly shown that the maximum of  $|H_2(j\lambda)|$  is decreased by increasing  $\delta$  and if  
 148  $\delta$  is sufficiently large, the frequency response of  $C2$  coincides with that of the TDVA. Such  
 149 an observation is also confirmed by other choices of  $\mu$ , as shown in Fig. 6. Therefore, it is  
 150 sufficient to conclude that for a single series arrangement of an inerter and a damper, the  
 151 series inerter provides no improvement for the  $H_\infty$  performance of the isolation system.

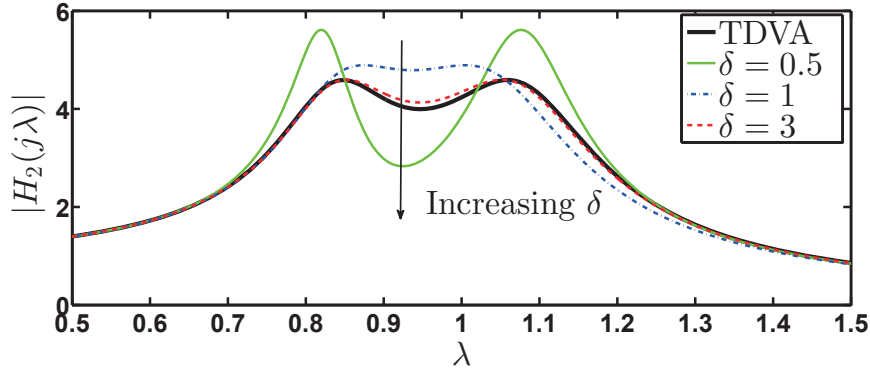


Figure 5: Comparison between the TDVA and  $C2$  when  $\mu = 0.1$  with different  $\delta$ .

152 The IDVAs  $C1$  and  $C2$  represent the two ways of adding an inerter to the TDVA, that is,  
 153 the parallel connection ( $C1$ ) and the series connection ( $C2$ ). Now, it has been demonstrated  
 154 that adding a single inerter alone to the TDVA, no matter it is in parallel connection or  
 155 in series connection, provides no improvement for the  $H_\infty$  performance. Therefore, other  
 156 degrees of freedom should be introduced, which is the motivation of introducing IDVAs  $C3$ ,  
 157  $C4$ ,  $C5$ , and  $C6$  by adding an inerter together with a spring to the TDVA.

### 158 3.2.2. Performance benefits of $C3$ , $C4$ , $C5$ , and $C6$

159 In this subsection, it will be shown that after adding another degree of freedom, that is  
 160 the spring  $k_1$ , the  $H_\infty$  performance will be significantly improved compared with the TDVA.

161 The optimization problem (8) with the constraint (10) is solved for configurations  $C3$ ,  
 162  $C4$ ,  $C5$  and  $C6$ , separately, where a 9th-order polynomial of equation (10) with respect to



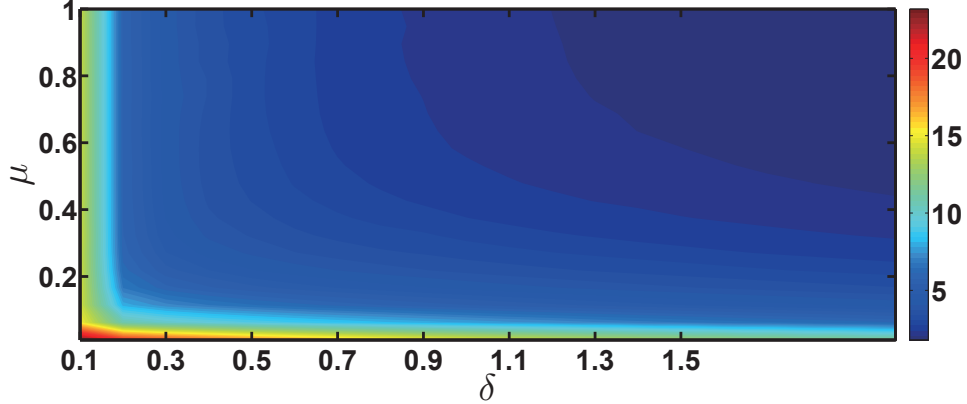


Figure 6:  $\max(|H_2(j\lambda)|)$  with different  $\mu$  and  $\delta$ .

163  $\lambda^2$  is obtained. The exact solutions of the TDVA in [4] are employed for comparison and the  
 164 detailed parameter values are shown in Tables 2, 3, and 4. Table 2 shows that all the IDVAs  
 165  $C3$ ,  $C4$ ,  $C5$  and  $C6$  can improve the  $H_\infty$  performance compared with the TDVA, where  $C3$   
 166 performs the best and the order of the performance is  $C3 > C6 > C4 > C5 > TDVA$  (“>”  
 167 means performing better) with an exception for  $\mu \geq 1$ . However, since the mass ratio is  
 168 normally quite small and practically less than 0.25 [42, 43], it is sufficient to conclude that  
 169  $C3 > C6 > C4 > C5 > TDVA$ . Such a conclusion is also confirmed by Fig. 7, where the  
 170 comparison of the IDVAs over the TDVA in the range of  $0 < \mu \leq 0.25$  is shown. As shown  
 171 in the right figure of Fig. 7, 8% to 26% improvement can be obtained for the IDVAs. The  
 172 other parameters in the range of  $0 < \mu \leq 0.25$  are depicted in Fig. 8. It should be noted  
 173 that although the optimal  $\gamma$  and  $\zeta$  for  $C3$  are almost identical to the TDVA, as shown in  
 174 Table 3 and Fig. 8, over 22% improvement can be provided by  $C3$  compared with the TDVA.  
 175 Moreover, the spring  $k_1$  is better to be in series connection for the  $H_\infty$  performance, given  
 176 the fact that  $C3$  and  $C6$  are superior to  $C4$  and  $C5$ .

177 The frequency responses of the IDVAs and the TDVA when  $\mu = 0.1$  are shown in Fig. 9,  
 178 where one sees that the magnitudes of the IDVAs around 1 are much flatter than those of  
 179 the TDVA, and the effective frequency band is much larger than that of the TDVA.

Table 2: maximum magnitude  $\max |H(j\lambda)|$  in the  $H_\infty$  optimization.

$\mu$	TDVA [4]	$C3$	$C4$	$C5$	$C6$
0.01	14.1796	11.0330	11.0860	12.9216	11.0351
0.02	10.0530	7.8340	7.9064	9.1498	7.8352
0.05	6.4080	5.0159	5.1194	5.8051	5.0210
0.1	4.5892	3.6175	3.7448	4.1379	3.6208
0.2	3.3254	2.6552	2.7986	2.9877	2.6616
0.5	2.2480	1.8513	1.9941	2.0198	1.8521
1	1.7457	1.4893	1.6127	1.5809	1.4893
2	1.4279	1.2697	1.3629	1.3157	1.2697
5	1.1942	1.1166	1.1702	1.1766	1.1166
10	1.1033	1.0602	1.0918	1.0934	1.0603

Table 3: Optimal natural frequency ratio  $\gamma$  and damping ratio  $\zeta$  in the  $H_\infty$  optimization.

(a) Optimal natural frequency ratio  $\gamma$

$\mu$	TDVA [4]	$C3$	$C4$	$C5$	$C6$
0.01	0.9902	0.9900	0.9957	0.9712	0.9842
0.02	0.9802	0.9802	0.9911	0.9493	0.9684
0.05	0.9520	0.9520	0.9766	0.9090	0.9242
0.1	0.9083	0.9083	0.9499	0.8501	0.8642
0.2	0.8319	0.8319	0.8931	0.7538	0.7693
0.5	0.6642	0.6643	0.7514	0.5681	0.5604
1	0.4973	0.4971	0.5882	0.4041	0.3979
2	0.3307	0.3302	0.4100	0.2547	0.2526
5	0.1646	0.1641	0.2145	0.2004	0.1197
10	0.0889	0.0893	0.1198	0.1118	0.0652

(b) Optimal damping ratio  $\zeta$

$\mu$	TDVA [4]	$C3$	$C4$	$C5$	$C6$
0.01	0.0603	0.0547	0.0025	0.0655	0.0025
0.02	0.0841	0.0769	0.0065	0.0973	0.0073
0.05	0.1276	0.1199	0.0224	0.1477	0.0270
0.1	0.1686	0.1657	0.0505	0.2086	0.0593
0.2	0.2101	0.2244	0.0981	0.2919	0.1180
0.5	0.2402	0.3175	0.2012	0.4294	0.3047
1	0.2235	0.3894	0.2905	0.5359	0.4354
2	0.1749	0.4505	0.3779	0.6325	0.5498
5	0.1002	0.5057	0.4525	0.5163	0.6593
10	0.0581	0.5288	0.4804	0.5313	0.6841

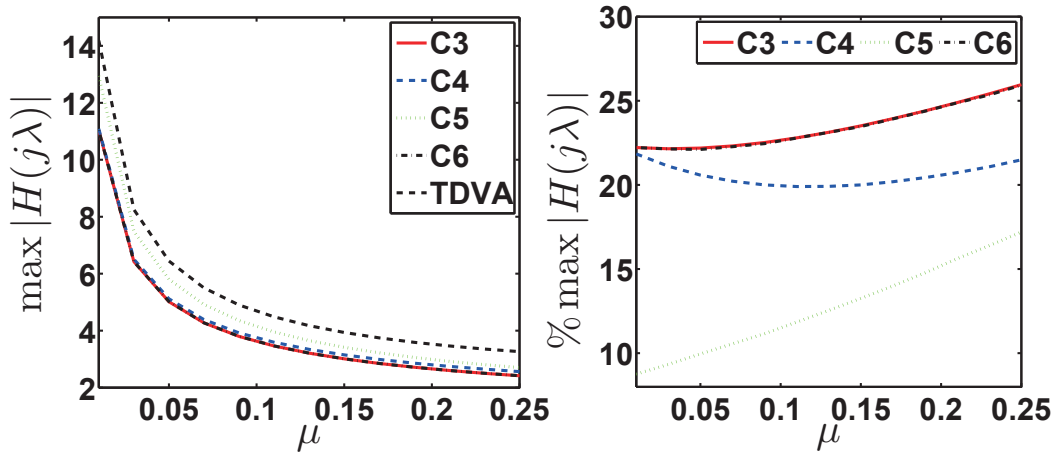


Figure 7: Maximum magnitude comparison between the IDVAs and the TDVA (left figure) and percentage improvement of the IDVAs with respect to the TDVA (right figure).

Table 4: Optimal inertance-to-mass ratio  $\delta$  and corner frequency ratio  $\eta$  in the  $H_\infty$  optimization.

(a) Optimal inertance-to-mass ratio $\delta$				
$\mu$	C3	C4	C5	C6
0.01	0.0238	0.0234	2.2791	0.0228
0.02	0.0473	0.0453	1.8105	0.0448
0.05	0.1156	0.1069	1.6782	0.0989
0.1	0.2208	0.1930	1.5320	0.1538
0.2	0.4082	0.3212	1.1521	0.2126
0.5	0.8256	0.5719	0.6919	0.2426
1	1.2552	0.7785	0.3130	0.2009
2	1.7228	0.9703	0.1423	0.1364
5	2.2540	1.1307	3.9018	0.0627
10	2.4989	1.2089	3.6257	0.0339

(b) Optimal corner frequency ratio $\eta$				
$\mu$	C3	C4	C5	C6
0.01	1.0051	0.9864	1.1242	1.0248
0.02	1.0098	0.9745	1.1982	1.0492
0.05	1.0248	0.9420	1.3341	1.1288
0.1	1.0485	0.9013	1.5181	1.2454
0.2	1.0940	0.8563	1.8754	1.4560
0.5	1.2219	0.7713	2.8856	2.2775
1	1.4061	0.7163	4.9686	3.5386
2	1.7178	0.6629	9.6074	6.0835
5	2.4169	0.6141	0.5009	14.5775
10	3.2632	0.5780	0.4739	27.6261

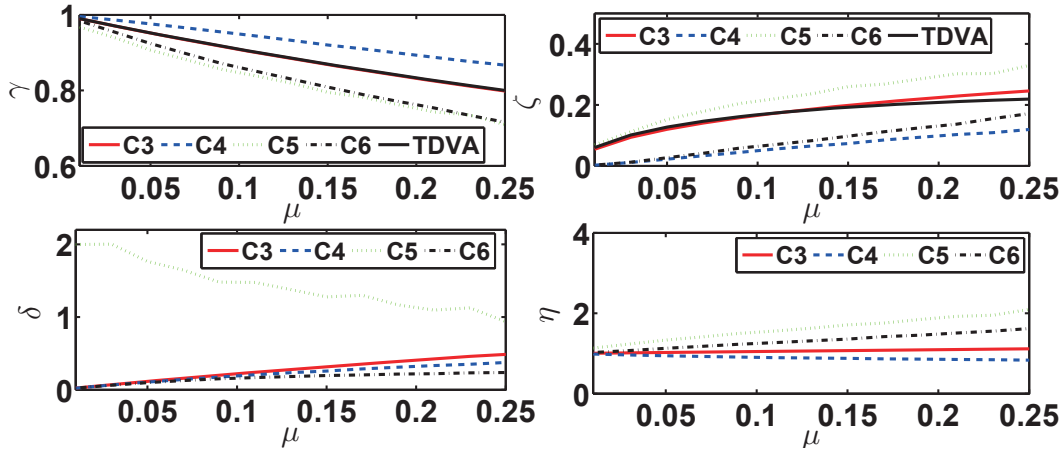


Figure 8: Optimal parameters in the  $H_\infty$  optimization: natural frequency ratio  $\gamma$  (up left); damping ratio  $\zeta$  (up right); inertance-to-mass ratio  $\delta$  (bottom left); corner frequency ratio  $\eta$  (bottom right).

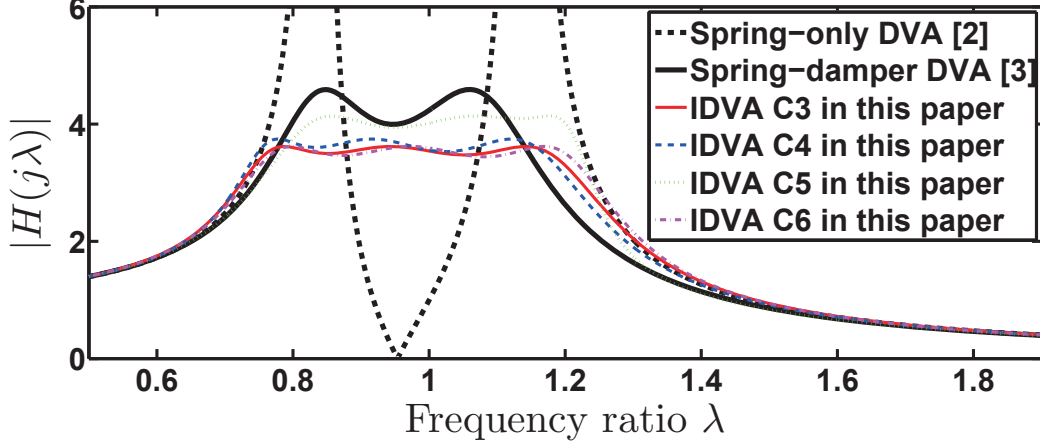


Figure 9: Comparison between the IDVAs and TDVA when  $\mu = 0.1$ . The spring-only DVA is the first DVA proposed by Frahm in 1909 [2]. The spring-damper DVA is the TDVA proposed by Ormondroyd and Den Hartog in 1928 [3].

## 180 4. $H_2$ optimization for the IDVAs

### 181 4.1. $H_2$ performance measure and its analytical solution

182 If the system is subjected to random excitation instead of sinusoidal excitation, the  $H_2$   
 183 optimization would be more desirable than the  $H_\infty$  optimization [9, 39, 40]. The performance  
 184 measure in the  $H_2$  optimization is defined as [9, 39, 40]

$$I = \frac{E[x^2]}{2\pi S_0 \omega_n}, \quad (11)$$

185 where  $S_0$  is the uniform power spectrum density function. The mean square value of  $x$  of the  
 186 object mass  $m$  can be calculated as

$$E[x^2] = S_0 \int_{-\infty}^{\infty} |H(j\lambda)|^2 d\omega = S_0 \omega_n \int_{-\infty}^{\infty} |H(j\lambda)|^2 d\lambda, \quad (12)$$

187 where  $H(j\lambda)$  is given in (7). Substituting (12) into (11), one obtains

$$I = \frac{1}{2\pi} \int_{-\infty}^{\infty} |H(j\lambda)|^2 d\lambda, \quad (13)$$

188 which is exactly the definition of the  $H_2$  norm of the transfer function  $\hat{H}(s)$  by replacing  $j\lambda$   
 189 in  $H(j\lambda)$  with the Laplace variable  $s$ .

190 Therefore, the  $H_2$  performance measure is rewritten as

$$I = \left\| \hat{H}(s) \right\|_2^2. \quad (14)$$

191 The analytical approach provided in [41, Chapter 2.6] will be employed to derive analytical  
 192 solutions for IDVAs in the  $H_2$  optimization, which is briefly presented as follows.

193 For a stable transfer function  $\hat{H}(s)$ , its  $H_2$  norm can be calculated as [41, Chapter 2.6]

$$\left\| \hat{H}(s) \right\|_2^2 = \|C(sI - A)^{-1}B\|_2^2 = CLC^T,$$

194 where  $A, B, C$  are the minimal state-space realization  $\hat{H}(s) = C(sI - A)^{-1}B$  and  $L$  is the  
 195 unique solution of the Lyapunov equation

$$AL + LA^T + BB^T = 0. \quad (15)$$

196 We can write  $\hat{H}(s)$

$$\hat{H}(s) = \frac{b_{n-1}s^{n-1} + \dots + b_1s + b_0}{s^n + a_{n-1}s^{n-1} + \dots + a_1s + a_0}$$

197 in its controllable canonical form below

$$\dot{x} = Ax + Bu, \quad y = Cx,$$

198 where

$$A = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ -a_0 & -a_1 & -a_2 & \dots & -a_{n-1} \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix}, \quad C = [b_0, b_1, b_2 \dots b_{n-1}].$$

#### 199 4.2. Comparison between the TDVA and IDVAs

200 For the TDVA, the  $H_2$  performance measure can be obtained as

$$I_{TDVA} = \frac{\gamma(1+\mu)\zeta}{\mu} + \frac{1 - (\mu+2)\gamma^2 + (1+\mu)^2\gamma^4}{4\mu\gamma\zeta}, \quad (16)$$

201 and the optimal  $\gamma$  and  $\zeta$  are

$$\gamma_{TDVA,opt} = \sqrt{\frac{\mu+2}{2(1+\mu)^2}}, \quad (17)$$

$$\zeta_{TDVA,opt} = \sqrt{\frac{(3\mu+4)\mu}{8(\mu+1)(\mu+2)}}. \quad (18)$$

202 Substituting  $\gamma_{TDVA,opt}$  and  $\zeta_{TDVA,opt}$  into (16), one obtains the optimal  $I_{TDVA,opt}$  as

$$I_{TDVA,opt} = \sqrt{\frac{3\mu+4}{4(\mu+1)\mu}}. \quad (19)$$

#### 203 4.2.1. Performance limitation of C1 and C2

204 The  $H_2$  performance measures for C1 and C2 can be obtained as

$$I_{C1} = \frac{\gamma(1+\mu)\zeta}{\mu} + \frac{1}{4\mu\gamma\zeta} (\delta^2 - 2((1+\mu)\gamma^2 - 1)\delta + 1 - (\mu+2)\gamma^2 + (1+\mu)^2\gamma^4) \quad (20)$$

$$= I_{TDVA} + \frac{1}{4\mu\gamma\zeta} (\delta^2 + a_{C1,1}\delta), \quad (21)$$

$$I_{C2} = (a_{C2,2}\delta^{-2} + a_{C2,1}\delta^{-1} + a_{C2,0})\zeta + \frac{1 - (\mu+2)\gamma^2 + (1+\mu)^2\gamma^4}{4\mu\gamma\zeta} \quad (22)$$

$$= I_{TDVA} + (a_{C2,2}\delta^{-2} + a_{C2,1}\delta^{-1})\zeta, \quad (23)$$

205 where

$$\begin{aligned}
a_{C1,1} &= -2((1 + \mu)\gamma^2 - 1), \\
a_{C2,2} &= \frac{\gamma}{\mu} ((1 + \mu)^3\gamma^4 - 2(1 + \mu)\gamma^2 + 1), \\
a_{C2,1} &= \frac{\gamma}{\mu} (2 + \mu - 2(1 + \mu)^2\gamma^2), \\
a_{C2,0} &= \frac{\gamma(1 + \mu)}{\mu}.
\end{aligned}$$

206 The following proposition can be obtained.

207 **Proposition 1.** *For the  $H_2$  performance,  $C1$  performs no better than the TDVA.*

208 *Proof.* See Appendix B. □

209 **Proposition 2.** *For the  $H_2$  performance,  $C2$  performs slightly better than the TDVA, but*  
210 *only at most 0.32% improvement can be achieved when  $\mu \leq 1$ .*

211 *Proof.* See Appendix C. □

212 Now, we have demonstrated that for the  $H_2$  performance,  $C1$  performs no better than  
213 the TDVA and  $C2$  provides negligible improvement over the TDVA. This means that adding  
214 an inerter alone to the TDVA provides limited improvement for the  $H_2$  performance, and  
215 therefore, another four IDVAs  $C3$ ,  $C4$ ,  $C5$ , and  $C6$  are proposed by adding an inerter together  
216 with a spring to the TDVA. It will be shown in the following sections that in this way, the  
217  $H_2$  performance can be significantly improved.

#### 218 4.2.2. Performance benefits of $C3$ , $C4$ , $C5$ , and $C6$

219 In this subsection, it will be analytically demonstrated that for the  $H_2$  performance,  
220 IDVAs  $C3$ ,  $C4$ ,  $C5$ , and  $C6$  perform surely better than the TDVA, and an optimization  
221 problem will be formulated to find the optimal parameters.

222 By using the method shown in Subsection 4.1, the analytical representations of the  $H_2$   
223 performance measures for  $C3$ ,  $C4$ ,  $C5$ , and  $C6$  are calculated and the detailed equations are  
224 shown in Appendix D. Denote the optimal  $H_2$  performances of  $C3$ ,  $C4$ ,  $C5$ , and  $C6$  as  
225  $I_{C3,opt}$ ,  $I_{C4,opt}$ ,  $I_{C5,opt}$ ,  $I_{C6,opt}$ , respectively. The following proposition can be obtained.

226 **Proposition 3.** *For the  $H_2$  performance, IDVAs  $C3$  and  $C5$  always perform better than the*  
227 *TDVA, that is, the following inequalities hold:*

$$I_{C3,opt} < I_{TDVA,opt}, \tag{24}$$

$$I_{C5,opt} < I_{TDVA,opt}, \tag{25}$$

228 *and if  $\mu \leq 1$ , IDVAs  $C4$  and  $C6$  always perform better than the TDVA, that is, the following*  
229 *inequalities hold:*

$$I_{C4,opt} < I_{TDVA,opt}, \tag{26}$$

$$I_{C6,opt} < I_{TDVA,opt}, \tag{27}$$

230 where  $I_{TDVA,opt}$  is the optimal  $H_2$  performance for the TDVA given by (19).

231 *Proof.* See Appendix E. □

232 **Remark 3.** *The condition  $\mu \leq 1$  for C4 and C6 in Proposition 3 is only a sufficient con-*  
 233 *dition, which means that for the case  $\mu > 1$ , it is also possible that the inequalities (26) and*  
 234 *(27) hold. However, such a condition introduces no conservativeness for DVA applications,*  
 235 *as the mass ratio  $\mu$  is normally less than 1 in practice (typically less than 0.25) [42, 43].*

236 *Since the IDVAs C3, C4, C5, C6 can always reduce to the TDVA by setting the spring*  
 237 *stiffness  $k_1$  (or  $\eta$ ) and inertance  $b$  (or  $\delta$ ) to 0 or  $\infty$ , the conclusions  $I_{C_i, \text{opt}} \leq I_{TDVA, \text{opt}}$ ,*  
 238  *$i = 3, 4, 5, 6$  always hold. However, Proposition 3 demonstrates the existence of finite  $\eta$  and*  
 239  *$\delta$  such that the IDVAs C3, C4, C5, and C6 are surely better than the TDVA.*

240 To determine the optimal values of  $\delta$ ,  $\gamma$ ,  $\eta$ , and  $\zeta$ , the following optimization problem  
 241 should be solved.

$$\min_{\delta, \gamma, \eta, \zeta} I_{C_i}, i = 3, 4, 5, 6, \quad (28)$$

242 subject to  $\delta > 0$ ,  $\gamma > 0$ ,  $\eta > 0$ , and  $\zeta > 0$ .

243 **Analytical solutions of C3:** Problem (28) can be analytically solved for C3, where the  
 244 optimal parameters for C3 are obtained as follows

$$\gamma_{C3, \text{opt}} = \sqrt{\frac{\sqrt{17\mu^2 + 32\mu + 16} - \mu}{4(1 + \mu)^2}}, \quad (29)$$

$$\eta_{C3, \text{opt}} = \sqrt{\frac{1 - 2(1 + \mu)\gamma_{C3, \text{opt}}^2 + (1 + \mu)\gamma_{C3, \text{opt}}^4}{(1 - (2 + 3\mu)\gamma_{C3, \text{opt}}^2 + (1 + \mu)^2\gamma_{C3, \text{opt}}^4)\gamma_{C3, \text{opt}}^2}}, \quad (30)$$

$$\delta_{C3, \text{opt}} = -\frac{2\hat{a}_{C3,2}}{\hat{a}_{C3,1}}, \quad (31)$$

$$\zeta_{C3, \text{opt}} = \sqrt{\frac{1 - (\mu + 2)\gamma_{C3, \text{opt}}^2 + (1 + \mu)^2\gamma_{C3, \text{opt}}^4}{4\mu\gamma_{C3, \text{opt}}(\hat{a}_{C3,2}\delta_{C3, \text{opt}}^{-2} + \hat{a}_{C3,1}\delta_{C3, \text{opt}}^{-1} + \hat{a}_{C3,0})}}, \quad (32)$$

245 where  $\hat{a}_{C3,2}$ ,  $\hat{a}_{C3,1}$ , and  $\hat{a}_{C3,0}$  are obtained by setting  $\gamma = \gamma_{C3, \text{opt}}$  and  $\eta = \eta_{C3, \text{opt}}$  for  $a_{C3,2}$ ,  
 246  $a_{C3,1}$ , and  $a_{C3,0}$ , respectively. For the representations of  $a_{C3,2}$ ,  $a_{C3,1}$ , and  $a_{C3,0}$ , see Appendix  
 247 D.

248 The analytical solutions  $\delta$ ,  $\gamma$ , and  $\eta$  are derived by successively setting the first derivatives  
 249 of  $I_{C3}$  with respect to  $\delta$ ,  $\eta$ , and  $\gamma$  as 0, and then checking the sign of the second derivatives  
 250 at stationary points. The optimal  $\zeta_{C3, \text{opt}}$  is derived due to the fact that both parts on the  
 251 right hand side of (D.1) of  $I_{C3}$  are positive.

252 **Solutions of C4, C5, and C6:** The analytical solutions of C4, C5, and C6 cannot be  
 253 obtained due to the high order equations (more than 4th-order) involved in the derivation.  
 254 However, the optimal solutions of  $\eta$  and  $\zeta$  can be analytically represented with respect to  $\delta$

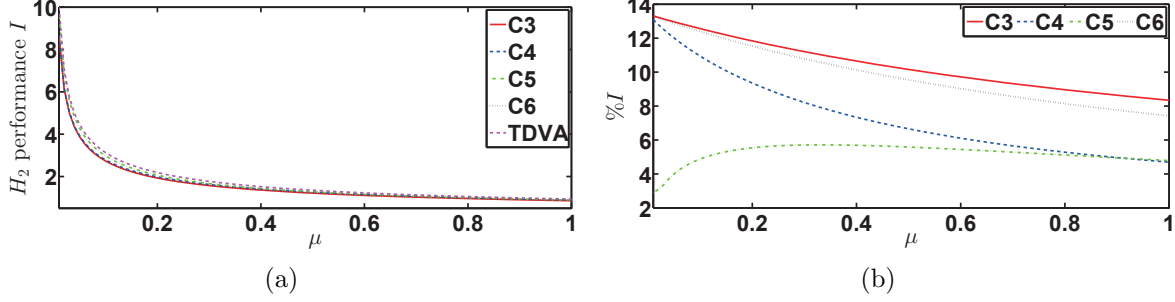


Figure 10: Comparison between IDVAs and the TDVA. (a) the  $H_2$  performance; (b) Percentage improvement of IDVAs with respect to the TDVA.

255 and  $\gamma$  as follows:

$$\eta_{C4,opt} = \frac{\sqrt{-(g_{C4,1}\delta + f_{C4,1})(2f_{C4,2} + 2g_{C4,2}\delta + 2l_{C4,2}\delta^2)}}{2(f_{C4,2} + g_{C4,2}\delta + l_{C4,2}\delta^2)}, \quad (33)$$

$$\zeta_{C4,opt} = \sqrt{\frac{l_{C4,2}\eta^4\delta^2 + l_{C4,1}\delta + l_{C4,0}}{a_{C4,2}\delta^{-2} + a_{C4,1}\delta^{-1} + a_{C4,0}}}, \quad (34)$$

$$\delta_{C5,opt} = -\frac{2a_{C5,2}}{a_{C5,1}}, \quad (35)$$

$$\zeta_{C5,opt} = \sqrt{\frac{1 - (\mu + 2)\gamma^2 + (1 + \mu)^2\gamma^4}{4\mu\gamma(a_{C5,2}\delta_{C5,opt}^{-2} + a_{C5,1}\delta_{C5,opt}^{-1} + a_{C5,0})}}, \quad (36)$$

$$\zeta_{C6,opt} = \sqrt{\frac{l_{C6,2}\eta^4\delta^2 + l_{C6,1}\delta + l_{C6,0}}{a_{C6,2}\delta^{-2} + a_{C6,1}\delta^{-1} + a_{C6,0}}}. \quad (37)$$

256 Correspondingly substituting the optimal representations above into  $I_{C_i}$ ,  $i = 4, 5, 6$ , the  
 257 problem (28) for  $C_i$ ,  $i = 4, 5, 6$  reduces to a nonlinear programming problem with two  
 258 unknown variables  $\delta$  and  $\gamma$  for  $C_4$  and  $C_5$ , and with three unknown variables  $\delta$ ,  $\gamma$  and  $\eta$  for  
 259  $C_6$ , which can be efficiently solved by using the Matlab solver *fmincon* and *GlobalSearch* in  
 260 Global Optimization Toolbox.

261 Fig. 10 and Fig. 11 depict the comparison between IDVAs  $C_3$ ,  $C_4$ ,  $C_5$ ,  $C_6$  and the  
 262 TDVA when  $0 \leq \mu \leq 1$ . As shown in Fig. 10(b),  $C_3$  performs the best, and more than 10%  
 263 improvement with respect to the TDVA can be obtained by  $C_3$ ,  $C_4$  and  $C_6$ . Similar to the  
 264  $H_\infty$  performance, the spring  $k_1$  is better to be in series connection for the  $H_2$  performance,  
 265 given the fact that  $C_3$  and  $C_6$  are superior to  $C_4$  and  $C_5$ .

## 266 5. Conclusions

267 In this paper, the performance of inerter-based dynamic vibration absorbers (IDVAs) has  
 268 been investigated, where the proposed IDVAs were a parallel arrangement of a spring and  
 269 an inerter-based mechanical network. Both  $H_\infty$  and  $H_2$  performances were considered. The  
 270  $H_\infty$  performance optimization problem was formulated in a minmax framework and solved by  
 271 using a direct search optimization method; while in the  $H_2$  optimization, an analytical method



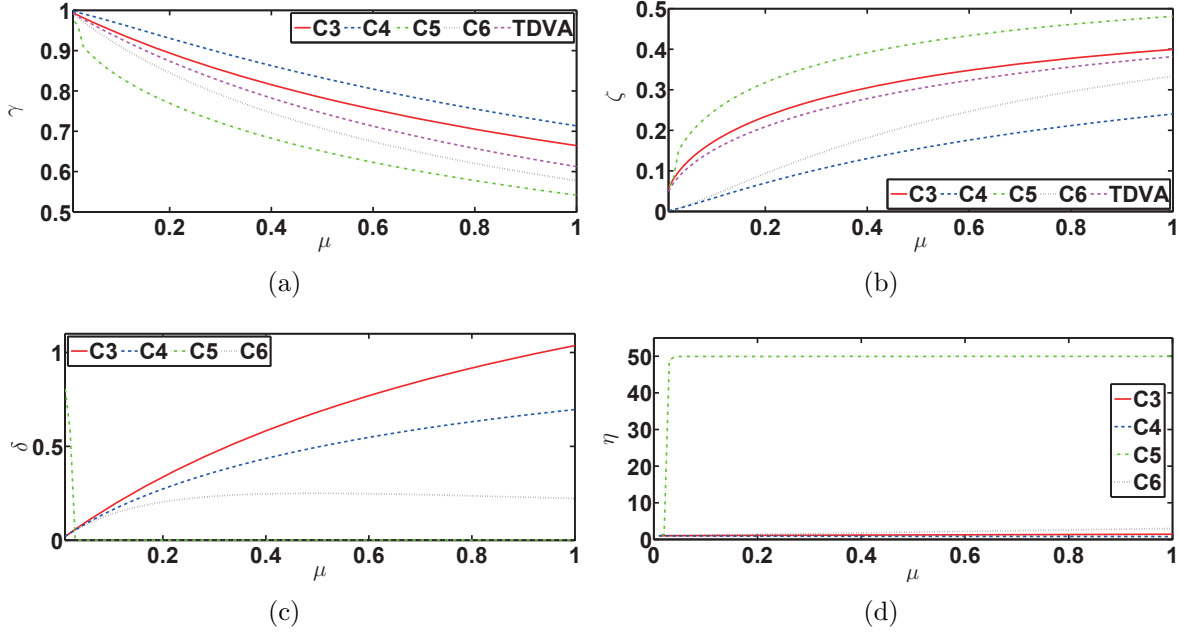


Figure 11: Optimal parameters: (a) optimal  $\gamma$ ; (b) optimal  $\zeta$ ; (c) optimal  $\delta$ ; (d) optimal  $\eta$ .

272 was employed to calculate the  $H_2$  performance measures. Comparisons between the proposed  
 273 IDVAs and the traditional dynamic vibration absorber (TDVA) were conducted. The results  
 274 showed that adding one inerter alone to the TDVA, no matter it is in parallel connection ( $C1$ )  
 275 or in series connection ( $C2$ ), provided no improvement for the  $H_\infty$  performance, and negligible  
 276 improvement (less than 0.32% improvement over the TDVA when the mass ratio less than  
 277 1) for the  $H_2$  performance. This demonstrated the necessity of introducing another degree of  
 278 freedom together with the inerter to the TDVA, and then the IDVAs  $C3$ ,  $C4$ ,  $C5$ , and  $C6$  were  
 279 proposed by adding an inerter together with a spring to the TDVA. Significant improvement  
 280 was obtained by IDVAs  $C3$ ,  $C4$ ,  $C5$ , and  $C6$ . For the  $H_\infty$  performance, numerical simulations  
 281 showed that over 20% improvement was achieved compared with the TDVA and the effective  
 282 frequency band can be enlarged by using inerter; while for the  $H_2$  performance, it was  
 283 analytically demonstrated that IDVAs  $C3$ ,  $C4$ ,  $C5$ , and  $C6$  were surely better than the  
 284 TDVA by carefully choosing the parameters, and over 10% improvement was obtained in the  
 285 numerical simulation.

286 **Appendix A. Detailed representations of  $R_{ni}$ ,  $I_{ni}$ ,  $R_{mi}$ , and  $I_{mi}$ ,  $i = 1, \dots, 6$ .**

$$\begin{aligned}
R_{n1} &= \lambda^2 - \gamma^2 + \delta\lambda^2, \\
I_{n1} &= -2\lambda\gamma\zeta, \\
R_{m1} &= (-\mu\delta - \delta - 1)\lambda^4 + (\gamma^2 + \mu\gamma^2 + 1 + \delta)\lambda^2 - \gamma^2, \\
I_{m1} &= 2\lambda\gamma\zeta(\lambda^2 - 1 + \mu\lambda^2), \\
R_{n2} &= \delta\lambda(\gamma^2 - \lambda^2), \\
I_{n2} &= -2\gamma\zeta(\gamma^2 - (1 + \delta)\lambda^2), \\
R_{m2} &= \delta\lambda(\lambda^4 - (\gamma^2 + \mu\gamma^2 + 1)\lambda^2 + \gamma^2), \\
I_{m2} &= -2\gamma\zeta((1 + \delta + \mu\delta)\lambda^4 - (\gamma^2 + \mu\gamma^2 + 1 + \delta)\lambda^2 + \gamma^2), \\
R_{n3} &= \delta\eta^2\gamma\lambda(\gamma^2 - \lambda^2), \\
I_{n3} &= -2\zeta(\gamma^4\eta^2 - (1 + \delta\eta^2 + \eta^2)\lambda^2\gamma^2 + \lambda^4), \\
R_{m3} &= \delta\eta^2\gamma\lambda(\lambda^4 - (1 + \gamma^2 + \mu\gamma^2)\lambda^2 + \gamma^2), \\
I_{m3} &= 2\zeta(\lambda^6 - (1 + \mu + \eta^2 + \delta\eta^2 + \mu\delta\eta^2)\lambda^4 + ((\mu + 1)\eta^2\gamma^2 + 1 + \eta^2 + \delta\eta^2)\gamma^2\lambda^2 - \gamma^4\eta^2), \\
R_{n4} &= -\delta(\lambda^4 - (1 + \eta^2 + \delta\eta^2)\gamma^2\lambda^2 + \gamma^4\eta^2), \\
I_{n4} &= -2\gamma\lambda\zeta(\gamma^2 - \lambda^2 - \delta\lambda^2), \\
R_{m4} &= \delta(\lambda^6 - (1 + (1 + \mu + \eta^2 + \delta\eta^2 + \delta\mu\eta^2)\gamma^2)\lambda^4 + ((\mu + 1)\eta^2\gamma^2 + (1 + \eta^2 + \delta\eta^2))\gamma^2\eta^2 - \gamma^4\eta^2), \\
I_{m4} &= -2\gamma\lambda\zeta((1 + \delta + \mu\delta)\lambda^4 - (1 + \delta + \gamma^2 + \mu\gamma^2)\lambda^2 + \gamma^2), \\
R_{n5} &= \delta(\gamma^2 - \lambda^2)(\lambda^2 - \eta^2\gamma^2), \\
I_{n5} &= -2\gamma\lambda\zeta((1 + \delta\eta^2)\gamma^2 - (1 + \delta)\lambda^2), \\
R_{m5} &= \delta(\lambda^2 - \eta^2\gamma^2)(\lambda^4 - (1 + \gamma^2 + \mu\gamma^2)\lambda^2 + \gamma^2), \\
I_{m5} &= -2\gamma\lambda\zeta((1 + \delta + \mu\delta)\lambda^4 - ((1 + \mu + \delta\eta^2 + \mu\delta\eta^2)\gamma^2 + 1 + \delta)\lambda^2 + (1 + \delta\eta^2)\gamma^2), \\
R_{n6} &= -\delta(\lambda^4 - (1 + \eta^2 + \delta\eta^2)\gamma^2\lambda^2 + \gamma^4\eta^2), \\
I_{n6} &= 2\lambda\gamma\zeta(\lambda^2 - (1 + \delta\eta^2)\gamma^2), \\
R_{m6} &= \delta(\lambda^6 - (1 + (1 + \mu + \eta^2 + \delta\eta^2 + \mu\delta\eta^2))\lambda^4 + ((\mu + 1)\eta^2\gamma^2 + (1 + \eta^2 + \delta\eta^2))\gamma^2\lambda^2 - \gamma^4\eta^2), \\
I_{m6} &= -2\gamma\lambda\zeta(\lambda^4 - (1 + (1 + \mu + \delta\eta^2 + \mu\delta\eta^2)\gamma^2)\lambda^2 + (1 + \delta\eta^2)\gamma^2).
\end{aligned}$$

287 **Appendix B. Proof of Proposition 1**

288 From (21), if  $C1$  performs better than the TDVA, that is  $I_{C1} < I_{TDVA}$ , the second term  
289 of (21) must be less than 0, which means

$$\delta^2 + a_{C1,1}\delta < 0.$$

290 Since  $\delta \geq 0$ , if  $\gamma^2 < \frac{1}{1+\mu}$ , the optimal  $\delta$  denoted as  $\delta_{opt}$  is 0. If  $\gamma^2 \geq \frac{1}{1+\mu}$ , the optimal  
291  $\delta_{opt} = (1 + \mu)\gamma^2 - 1$ , and it can be checked that the optimal  $\gamma$  is  $\frac{1}{1+\mu}$  by substituting  $\delta_{opt}$  into  
292 (21), which means that the optimal  $\delta$  is also 0.

293 **Appendix C. Proof of Proposition 2**

294 First, we prove that  $C2$  performs better than the TDVA, that is  $I_{C2,opt} < I_{TDVA,opt}$ ,  
 295 where  $I_{C2,opt}$  denotes the optimal  $I_{C2}$ . From (23), if  $C2$  performs better than the TDVA, the  
 296 following inequality must hold

$$a_{C2,2}\delta^{-2} + a_{C2,1}\delta^{-1} < 0,$$

297 which requires that

$$a_{C2,1} < 0 \text{ or } \gamma^2 > \frac{2 + \mu}{2(1 + \mu)^2},$$

298 as  $a_{C2,2} \geq 0$  for any  $\gamma \geq 0$ . If  $\gamma^2 > \frac{2+\mu}{2(1+\mu)^2}$ , the optimal  $\delta^{-1}$  is

$$\delta_{opt}^{-1} = -\frac{a_{C2,1}}{2a_{C2,2}},$$

299 and  $I_{C2}$  can be represented as

$$I_{C2} = \sqrt{\frac{(1 - (2 + \mu)\gamma^2 + (1 + \mu)^2\gamma^4)(4(1 + \mu)^2\gamma^2 - \mu)}{4\mu(1 - 2(1 + \mu)\gamma^2 + (1 + \mu)^3\gamma^4)}}. \quad (C.1)$$

300 Using  $I_{TDVA,opt}$  given in (16), one obtains

$$I_{C2}^2 - I_{TDVA,opt}^2 = \frac{((\mu + 1)\gamma^2 - 1)(2(\mu + 1)^2\gamma^2 - 2 - \mu)^2}{4\mu(1 - 2(\mu + 1)\gamma^2 + (\mu + 1)^3\gamma^4)(\mu + 1)},$$

301 Clearly, if  $\gamma^2 < \frac{1}{1+\mu}$ , then  $I_{C2} < I_{TDVA,opt}$ . Since  $\frac{1}{1+\mu} > \frac{2+\mu}{2(1+\mu)^2}$ , one can always find a  $\gamma$   
 302 such that  $I_{C2} < I_{TDVA,opt}$ . Since  $I_{C2,opt} \leq I_{C2}$ , one obtains  $I_{C2,opt} < I_{TDVA,opt}$ .

303 Second, we graphically prove that only at most 0.32% improvement can be obtained by  
 304  $C2$  when  $\mu \leq 1$ . The optimal  $\gamma$  can be obtained by solving  $\frac{\partial I_{C2}^2}{\partial \gamma^2} = 0$ , which is equivalent to

$$(2\alpha^2\gamma^2 - 1 - \alpha)(2\alpha^5\gamma^6 + (\alpha^4 - 7\alpha^3)\gamma^4 + (8\alpha^2 - 2\alpha^3)\gamma^2 - 3\alpha + 1) = 0, \quad (C.2)$$

305 where  $\alpha = \mu + 1$ . It is easy to check that (C.2) has two real positive solutions denoted as  $\gamma_1$   
 306 and  $\gamma_2$ ,  $\gamma_1 < \gamma_2$ , where

$$\gamma_1 = \sqrt{\frac{1 + \alpha}{2\alpha^2}},$$

307 and  $\gamma_1 < \gamma_2 < \sqrt{2}\gamma_1$ . Also,  $\gamma_2^2$  is the unique real solution of equation

$$2\alpha^5\gamma^6 + (\alpha^4 - 7\alpha^3)\gamma^4 + (8\alpha^2 - 2\alpha^3)\gamma^2 - 3\alpha + 1 = 0,$$

308 and the optimal  $\gamma$  is  $\gamma_2$ .

309 For  $0 \leq \mu \leq 1$ , a graphical comparison with the TDVA is shown in Fig. C.12, where it is  
 310 clearly shown that at most 0.32% improvement is obtained for  $C2$ .

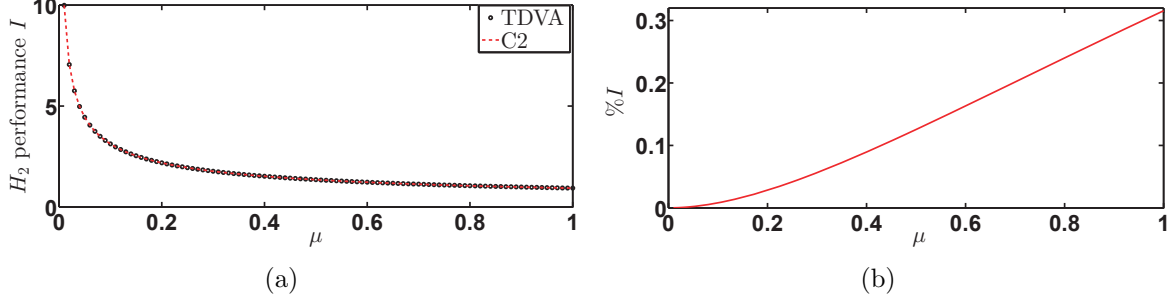


Figure C.12: Comparison between  $C2$  and TDVA when  $0 \leq \mu \leq 1$ . (a) the  $H_2$  performance; (b) Percentage improvement of  $C2$  with respect to TDVA.

311 **Appendix D. Analytical representations of the  $H_2$  performance measures for**  
 312  **$C3$ ,  $C4$ ,  $C5$ , and  $C6$**

313 Denote  $I_{C3}$ ,  $I_{C4}$ ,  $I_{C5}$ , and  $I_{C6}$  as the  $H_2$  performance measures for  $C3$ ,  $C4$ ,  $C5$ , and  $C6$ ,  
 314 respectively. The detailed representations are obtained as follows:

$$\begin{aligned}
 I_{C3} &= (a_{C3,2}\delta^{-2} + a_{C3,1}\delta^{-1} + a_{C3,0}) \zeta + \frac{1 - (\mu + 2)\gamma^2 + (1 + \mu)^2\gamma^4}{4\gamma\mu\zeta} \\
 &= I_{TDVA} + (a_{C3,2}\delta^{-2} + a_{C3,1}\delta^{-1}) \zeta, \tag{D.1}
 \end{aligned}$$

$$\begin{aligned}
 I_{C4} &= (a_{C4,2}\delta^{-2} + a_{C4,1}\delta^{-1} + a_{C4,0}) \zeta + (l_{C4,2}\eta^4\delta^2 + l_{C4,1}\delta + l_{C4,0}) \frac{1}{\zeta} \\
 &= I_{TDVA} + (a_{C4,2}\delta^{-2} + a_{C4,1}\delta^{-1}) \zeta + (l_{C4,2}\eta^4\delta^2 + l_{C4,1}\delta + f_{C4,2}\eta^4 + f_{C4,1}\eta^2) \frac{1}{\zeta},
 \end{aligned}$$

$$\begin{aligned}
 I_{C5} &= (a_{C5,2}\delta^{-2} + a_{C5,1}\delta^{-1} + a_{C5,0}) \zeta + \frac{1}{4\gamma\mu\zeta} (1 - (\mu + 2)\gamma^2 + (1 + \mu)^2\gamma^4) \\
 &= I_{TDVA} + (a_{C5,2}\delta^{-2} + a_{C5,1}\delta^{-1}) \zeta, \tag{D.2}
 \end{aligned}$$

$$\begin{aligned}
 I_{C6} &= (a_{C6,2}\delta^{-2}\eta^{-4} + a_{C6,1}\delta^{-1}\eta^{-2} + a_{C6,0}) \zeta + (l_{C6,2}\delta^2 + l_{C6,1}\delta + l_{C6,0}) \frac{1}{\zeta} \\
 &= I_{TDVA} + (a_{C6,2}\delta^{-2}\eta^{-4} + a_{C6,1}\delta^{-1}\eta^{-2}) \zeta + (l_{C6,2}\delta^2 + l_{C6,1}\delta + f_{C6,2}\eta^{-4} + f_{C6,1}\eta^{-2}) \frac{1}{\zeta},
 \end{aligned}$$

315 where

$$\begin{aligned}
a_{C3,2} &= d_{C3,2}\eta^{-4} + d_{C3,1}\eta^{-2} + d_{C3,0}, \quad a_{C3,1} = g_{C3,1}\eta^{-2} + g_{C3,0}, \quad a_{C3,0} = \frac{\gamma(1+\mu)}{\mu}, \\
d_{C3,2} &= \frac{1}{\gamma^3\mu} (1 - 2\gamma^2 + (1+\mu)\gamma^4), \quad d_{C3,1} = -\frac{2}{\gamma\mu} (1 - (2+\mu)\gamma^2 + (1+\mu)^2\gamma^4), \\
d_{C3,0} &= \frac{\gamma}{\mu} (1 - 2(1+\mu)\gamma^2 + (1+\mu)^3\gamma^4), \quad g_{C3,1} = -\frac{2}{\mu\gamma} (1 - (1+\mu)\gamma^2), \\
g_{C3,0} &= -\frac{\gamma}{\mu} (2(1+\mu)^2\gamma^2 - 2 - \mu), \\
a_{C4,2} &= \frac{\gamma}{\mu} (1 - (2+\mu)\gamma^2 + (1+\mu)^3\gamma^4), \quad a_{C4,1} = \frac{\gamma}{\mu} (2+\mu - 2(1+\mu)^2\gamma^2), \quad a_{C4,0} = \frac{\gamma(1+\mu)}{\mu}, \\
l_{C4,2} &= \frac{\gamma^3(1+\mu)^2}{4\mu}, \quad l_{C4,1} = g_{C4,2}\eta^4 + g_{C4,1}\eta^2, \quad l_{C4,0} = f_{C4,2}\eta^4 + f_{C4,1}\eta^2 + f_{C4,0}, \\
g_{C4,2} &= \frac{\gamma^3}{2\mu} (1+\mu - (1+\mu)^3\gamma^2), \quad g_{C4,1} = \frac{\gamma}{4\mu} (2(1+\mu)^2\gamma^2 - \mu - 2), \\
f_{C4,2} &= \frac{\gamma^3}{4\mu} ((1+\mu)^4\gamma^4 + (\mu-2)(\mu+1)^2\gamma^2 + 1), \quad f_{C4,1} = -\frac{\gamma}{2\mu} ((1+\mu)^3\gamma^4 - 2(1+\mu)\gamma^2 + 1), \\
f_{C4,0} &= \frac{1}{4\mu\gamma} (1 - (\mu+2)\gamma^2 + (1+\mu)^2\gamma^4), \\
a_{C5,2} &= \frac{g_{C5,2}\eta^4 + g_{C5,1}\eta^2 + g_{C5,0}}{\mu(1 + f_{C5,1}\eta^2 + f_{C5,2}\eta^4)^2}, \quad a_{C5,1} = \frac{l_{C5,3}\eta^6 + l_{C5,2}\eta^4 + l_{C5,1}\eta^2 + l_{C5,0}}{\mu(1 + f_{C5,1}\eta^2 + f_{C5,2}\eta^4)^2}, \quad a_{C5,0} = \frac{\gamma(1+\mu)}{\mu}, \\
g_{C5,2} &= \gamma((1+\mu)\gamma^4 - 2\gamma^2 + 1), \quad g_{C5,1} = -2\gamma((1+\mu)^2\gamma^4 - (\mu+2)\gamma^2 + 1), \\
g_{C5,0} &= \gamma((1+\mu)^3\gamma^4 - 2(1+\mu)\gamma^2 + 1), \quad f_{C5,1} = -(1 + \gamma^2(1+\mu)), \quad f_{C5,2} = \gamma^2, \\
l_{C5,3} &= 2\gamma^3((1+\mu)^3 - 1), \quad l_{C5,2} = -\gamma(4(1+\mu)^2\gamma^4 - 2\gamma^2 - \mu - 2), \\
l_{C5,1} &= 2\gamma((1+\mu)^3\gamma^4 + (1+\mu)^2\gamma^2 - \mu - 2), \quad l_{C5,0} = \gamma(\mu + 2 - 2(1+\mu)^2\gamma^2), \\
a_{C6,2} &= \frac{1 - 2\gamma^2 + (1+\mu)\gamma^4}{\gamma^3\mu}, \quad a_{C6,1} = \frac{2((1+\mu)\gamma^2 - 1)}{\gamma\mu}, \quad a_{C6,0} = \frac{\gamma(1+\mu)}{\mu}, \\
l_{C6,2} &= \frac{1}{4\gamma\mu}, \quad l_{C6,1} = g_{C6,1}\eta^{-2} + g_{C6,0}, \quad l_{C6,0} = f_{C6,2}\eta^{-4} + f_{C6,1}\eta^{-2} + f_{C6,0}, \\
g_{C6,1} &= \frac{\mu - 2 + 2\gamma^2}{4\gamma^3\mu}, \quad g_{C6,0} = \frac{1 - (1+\mu)\gamma^2}{2\gamma\mu}, \\
f_{C6,2} &= \frac{1 + (\mu-2)\gamma^2 + \gamma^4}{4\mu\gamma^5}, \quad f_{C6,1} = -\frac{1 - 2\gamma^2 + (1+\mu)\gamma^4}{2\mu\gamma^3}, \quad f_{C6,0} = \frac{1 - (2+\mu)\gamma^2 + (1+\mu)^2\gamma^4}{4\gamma\mu}.
\end{aligned}$$

316 **Appendix E. Proof of Proposition 3**

317 For  $C3$ , substituting  $\gamma_{TDVA,opt}$  and  $\zeta_{TDVA,opt}$  into (D.1), one obtains

$$I'_{C3} = I_{TDVA,opt} + (a'_{C3,2}\delta^{-2} + a'_{C3,1}\delta^{-1}) \zeta_{TDVA,opt},$$

318 where  $a'_{C3,2}$  and  $a'_{C3,1}$  are obtained by setting  $\gamma = \gamma_{TDVA,opt}$  for  $a_{C3,2}$  and  $a_{C3,1}$ , respectively.  
 319 It can be checked that  $a'_{C3,2} > 0$  and

$$a'_{C3,1} = -\sqrt{\frac{2}{2+\mu}}\eta^{-2} < 0,$$

320 which means that there exist finite  $\delta$  and  $\eta$  such that  $I'_{C3} < I_{TDVA,opt}$ . Since  $I_{C3,opt} \leq I'_{C3}$ ,  
 321 then one obtains  $I_{C3,opt} < I_{TDVA,opt}$ .

322 For  $C4$ , denote

$$I'_{C4} = 2\sqrt{(a'_{C4,2}\delta^{-2} + a'_{C4,1}\delta^{-1} + a'_{C4,0})(l'_{C4,2}\eta^4\delta^2 + l'_{C4,1}\delta + l'_{C4,0})},$$

323 where  $a'_{C4,2}$ ,  $a'_{C4,1}$ ,  $a'_{C4,0}$ ,  $l'_{C4,2}$ ,  $l'_{C4,1}$ , and  $l'_{C4,0}$  are obtained by setting  $\gamma = \gamma_{TDVA,opt}$ .

324 Expanding  $I'_{C4}$ , one obtains

$$I'_{C4} = 2\sqrt{a'_{C4,0}f'_{C4,0} + f_{C4,\eta}}, \quad (\text{E.1})$$

325 where

$$f_{C4,\eta} = (l'_{C4,2}\delta^2 + g'_{C4,2}\delta + f'_{C4,2})(a'_{C4,2}\delta^{-2} + a'_{C4,0})\eta^4 + f'_{C4,1}(a'_{C4,2}\delta^{-2} + a'_{C4,0})\eta^2 + f'_{C4,0}a'_{C4,2}\delta^{-2}.$$

326 Note that

$$I_{TDVA,opt} = 2\sqrt{a'_{C4,0}f'_{C4,0}}.$$

327 Then, we will prove that there exist finite  $\delta$  and  $\eta$  so that  $f_{C4,\eta} < 0$ . It can be checked  
 328 that  $l'_{C4,2}\delta^2 + g'_{C4,2}\delta + f'_{C4,2} > 0$ ,  $a'_{C4,2}\delta^{-2} + a'_{C4,0} > 0$ , and  $f'_{C4,1}(a'_{C4,2}\delta^{-2} + a'_{C4,0}) < 0$ . The  
 329 discriminant of  $f_{C4,\eta} = 0$  is

$$\Delta = (a'_{C4,2}\delta^2 + a'_{C4,0}) \left( (f'_{C4,1})^2 - 4f'_{C4,2}f'_{C4,0} \right) a'_{C4,2}\delta^{-2} - 4g'_{C4,2}f'_{C4,0}a'_{C4,2}\delta^{-1} +$$

$$f'_{C4,1}{}^2 a'_{C4,0} - 4l'_{C4,2}f'_{C4,0}a'_{C4,2}.$$

330  
 331 It can be checked that if  $\mu < \frac{8\sqrt{2}-4}{7} \approx 1.045$ , there exists a finite  $\delta$  such that the second term  
 332 of  $\Delta$  is positive, which means that if  $\mu < 1.045$ , there exists a finite  $\eta$  such that  $f_{C4,\eta} < 0$ .  
 333 For example, if choosing

$$\delta^{-1} = \frac{2g'_{C4,2}f'_{C4,0}}{f'_{C4,1}{}^2 - 4f'_{C4,2}f'_{C4,0}} = \frac{(3\mu+4)(1+\mu)}{4\mu(\mu+2)}, \quad (\text{E.2})$$

334 and

$$\eta = \sqrt{\frac{-f'_{C4,1}}{l'_{C4,2}\delta^2 + g'_{C4,2}\delta + f'_{C4,2}}} = \sqrt{\frac{2(3\mu+4)^2(1+\mu)(4+\mu)}{(\mu+2)(43\mu^3 + 204\mu^2 + 272\mu + 64)}}, \quad (\text{E.3})$$

335 one obtains

$$f_{C4,\eta} = \frac{1}{128} \frac{(7\mu^2 + 8\mu - 16)(\mu + 4)(3\mu + 4)^2}{\mu(43\mu^3 + 204\mu^2 + 272\mu + 64)(1 + \mu)(\mu + 2)} < 0.$$

336 From (E.1) and for the  $\delta$  and  $\eta$  given by (E.2) and (E.3), one obtains that if  $\mu < 1.045$ ,

$$I'_{C4} < I_{TDVA,opt}.$$

337 Since  $I_{C4,opt} \leq I'_{C4}$ , one obtains that if  $\mu < 1.045$ ,  $I_{C4,opt} < I_{TDVA,opt}$ .

338 For  $C5$ , setting  $\gamma = \gamma_{TDVA,opt}$  and  $\zeta = \zeta_{TDVA,opt}$  in (D.2), one obtains

$$I'_{C5} = I_{TDVA,opt} + (a'_{C5,2}\delta^{-2} + a'_{C5,1}\delta^{-1})\zeta_{TDVA,opt}. \quad (E.4)$$

339 Then, we will show that there exist finite  $\delta$  and  $\eta$  such that  $a'_{C5,2}\delta^{-2} + a'_{C5,1}\delta^{-1} < 0$ . It can  
340 be checked that  $a'_{C5,2} > 0$ . Therefore, we only need to prove that there exists a finite  $\eta$  such  
341 that  $a'_{C5,1} < 0$ . Since

$$a'_{C5,1} = \frac{l'_{C5,3}\eta^6 + l'_{C5,2}\eta^4 + l'_{C5,1}\eta^2}{\mu(1 + f'_{C5,1}\eta^2 + f'_{C5,2}\eta^4)^2},$$

342 it is easy to check that  $a'_{C5,1} < 0$  if  $\eta^2 > (\mu + 1)(\mu + 1 + \sqrt{\mu^2 + 2\mu})$  or  $\eta^2 < (\mu +$   
343  $1)(\mu + 1 - \sqrt{\mu^2 + 2\mu})$ . For example, if choosing

$$\eta = \sqrt{2(1 + \mu)^2}, \quad (E.5)$$

$$\delta^{-1} = \frac{2(2 + \mu)(\mu + 1)^2}{(1 + 8\mu + 4\mu^2)(4 + 9\mu + 4\mu^2)}, \quad (E.6)$$

344 one obtains

$$f_\delta = -\frac{\sqrt{2}(2 + \mu)^{5/2}(\mu + 1)^2}{(1 + 8\mu + 4\mu^2)(4 + 9\mu + 4\mu^2)(1 + 3\mu + 5\mu^2 + 2\mu^3)^2} < 0,$$

345 which means that for the  $\eta$  and  $\delta$  given by (E.5) and (E.6),  $I'_{C5} < I_{TDVA,opt}$ . Since  $I_{C5,opt} \leq$   
346  $I'_{C5}$ , one obtains  $I_{C5,opt} < I_{TDVA,opt}$ .

347 For  $C6$ , setting  $\gamma = \gamma_{TDVA,opt}$  and  $\zeta = \zeta_{TDVA,opt}$ , one obtains

$$I'_{C6} = I_{TDVA,opt} + f_{C6,\eta},$$

348 where  $f_{C6,\eta} = d_2\eta^{-4} + d_1\eta^{-2} + d_0$ , with

$$\begin{aligned} d_2 &= a'_{C6,2}\zeta_{TDVA,opt}\delta^{-2} + f'_{C6,2}/\zeta_{TDVA,opt}, \\ d_1 &= a'_{C6,1}\zeta_{TDVA,opt}\delta^{-1} + (g'_{C6,1}\delta + f'_{C6,1})/\zeta_{TDVA,opt}, \\ d_0 &= (l'_{C6,2}\delta^2 + g'_{C6,0}\delta)/\zeta_{TDVA,opt}. \end{aligned}$$

349 It can be checked that  $d_2 > 0$  for any  $\delta$  and if  $\mu < \sqrt{2}$ ,  $d_1 < 0$ . Thus, it remains to  
350 prove that there exists a finite  $\eta > 0$  such that  $f_{C6,\eta} < 0$ . This can be done by checking the  
351 discriminant of  $f_{C6,\eta}$ , which is

$$\begin{aligned} \Delta &= d_1^2 - 4d_2d_0 \\ &= 16(\mu - 4)(\mu + 1)^8\delta^4 - 16\mu(4\mu^3 + 11\mu^2 + 5\mu - 4)(\mu + 1)^4\delta^3 + \\ &\quad 8\mu^2(5\mu^2 + 21\mu + 20)(\mu + 1)^3\delta^2 + \mu^3(3\mu + 4)^2. \end{aligned}$$

352 It is easy to see that there always exists a finite  $\delta$  such that  $\Delta > 0$ . For example, if choosing

$$\delta = \frac{\mu(4\mu^3 + 11\mu^2 + 5\mu - 4 - \sqrt{6\mu^6 + 56\mu^5 + 253\mu^4 + 606\mu^3 + 799\mu^2 + 568\mu + 176})}{2(\mu - 4)(\mu + 1)^4},$$

353 which is larger than 0 if  $\mu < 4$ , one obtains

$$\Delta = \mu^3(3\mu + 4)^2 > 0.$$

354 Therefore, we can always find a  $\eta^{-2}$  between the two real positive solutions of  $f_{C6,\eta} = 0$  such  
355 that  $f_{C6,\eta} < 0$ . A possible choice is  $\eta^{-2} = -\frac{d_1}{2d_2}$ . This means that if carefully choosing  $\delta$  and  
356  $\eta$ , the inequality  $I'_{C6} < I_{IDVA,opt}$  holds. Since  $I_{C6,opt} \leq I'_{C6}$ , one obtains  $I_{C6,opt} < I_{TDVA,opt}$ .

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### 364 **References**

- 365 [1] Den Hartog JP. Mechanical Vibrations. New York: Courier Dover Publications; 1985.
- 366 [2] Frahm H. Device for damping vibrations of bodies. US Patent, No. 989958. 30 October  
367 1909.
- 368 [3] Ormondroyd J, Den Hartog JP. The theory of the dynamic vibration absorber. ASME  
369 Journal of Applied Mechanics 1928;50:9–22.
- 370 [4] Nishihara O, Asami T. Closed-form solutions to the exact optimizations of dynamic  
371 vibration absorbers (minimizations of the maximum amplitude magnification factors).  
372 Journal of Vibration and Acoustics 2002;124(4):576–82.
- 373 [5] Crandall SH, Mark WD. Random Vibration in Mechanical Systems. New York: Aca-  
374 demic Press; 1963.
- 375 [6] Asami T, Wakasono T, Kameoka K, Hasegawa M, Sekiguchi H. Optimum design of  
376 dynamic absorbers for a system subjected to random excitation. JSME International  
377 Journal Ser. 3, Vibration, Control Engineering, Engineering for Industry 1991;34(2):218-  
378 26.
- 379 [7] Anh ND, Nguyen NX. Design of TMD for damped linear structures using the dual  
380 criterion of equivalent linearization method. International Journal of Mechanical Sciences  
381 2013;77:164–70.



- 382 [8] Ghosh A, Basu B. A closed-form optimal tuning criterion for TMD in damped structures.  
383 Structural Control and Health Monitoring 2007;14:681–92.
- 384 [9] Asami T, Nishihara O, Baz AM. Analytical solutions to  $H_\infty$  and  $H_2$  optimization of  
385 dynamic vibration absorbers attached to damped linear systems. Journal of Vibration  
386 and Acoustics 2002;124(2):284–95.
- 387 [10] Bekdas G, Nigdeli SM. Mass ratio factor for optimum tuned mass damper strategies.  
388 International Journal of Mechanical Sciences 2013;71:68-84.
- 389 [11] Cheung YL, Wong WO.  $H_\infty$  and  $H_2$  optimizations of a dynamic vibration absorber for  
390 suppressing vibrations in plates. Journal of Sound and Vibration 2009;320:29–42.
- 391 [12] Pai PF, Schulz MJ. A refined nonlinear vibration absorber. International Journal of  
392 Mechanical Sciences 2000;42(3):537–60.
- 393 [13] Miguelez MH, Rubio L, Loya JA, Fernandez-Saez J. Improvement of chatter stability in  
394 boring operations with passive vibration absorbers. International Journal of Mechanical  
395 Sciences 2010;52(10):1376–84.
- 396 [14] Gao H, Zhan W, Karimi HR, Yang X, Yin S. Allocation of actuators and sensors for  
397 coupled-adjacent-building vibration attenuation. IEEE Transactions on Industrial Elec-  
398 tronics 2013;60(12):5792–801.
- 399 [15] Si Y, Karimi HR, Gao H. Modelling and optimization of a passive structural control  
400 design for a spar-type floating wind turbine. Engineering Structures 2014;69:168–82.
- 401 [16] Zhan W, Cui Y, Feng Z, Cheung KC, Lam J, Gao H. Joint optimization approach  
402 to building vibration control via multiple active tuned mass dampers. Mechatronics  
403 2013;23(3):355–68.
- 404 [17] Smith MC. Synthesis of mechanical networks: the inerter. IEEE Trans Autom Control  
405 2002;47(10):1648–62.
- 406 [18] Chen MZQ, Papageorgiou C, Scheibe F, Wang FC, Smith MC. The missing mechanical  
407 circuit element. IEEE Circuits Syst Mag 2009;9(1):10–26.
- 408 [19] Smith MC, Wang FC. Performance benefits in passive vehicle suspensions employing  
409 inerters. Veh Syst Dyn 2004;42(4):235–57.
- 410 [20] Chen MZQ, Hu Y, Du B. Suspension performance with one damper and one inerter. The  
411 24th Chinese Control and Decision Conference. Taiyuan: China;2012, p. 3551–3556.
- 412 [21] Chen MZQ, Hu Y, Li C, Chen G. Performance benefits of using inerter in semiactive sus-  
413 pensions. IEEE Trans Control Syst Technol, in press. DOI:10.1109/TCST.2014.2364954.
- 414 [22] Hu Y, Chen MZQ, Shu Z. Passive vehicle suspensions employing inerters with multiple  
415 performance requirements. Journal of Sound and Vibration 2014;333(8):2212–25.

- 416 [23] Lazar IF, Neild SA, Wagg DJ. Using an inerter-based device for structural vibration  
417 suppression. *Earthquake Engng Struct Dyn* 2014;43(8):1129–47.
- 418 [24] Hu Y, Chen MZQ, Shu Z, Huang L. Analysis and optimisation for inerter-based iso-  
419 lators via fixed-point theory and algebraic solution. *Journal of Sound and Vibration*  
420 2015;346:17–36.
- 421 [25] Marian L, Giaralis, A. Optimal design of a novel tuned mass-damper-inerter(TMDI)  
422 passive vibration control configuration for stochastically support-excited structural sys-  
423 tems. *Probabilistic Engineering Mechanics* 2014;38:156–64.
- 424 [26] Brzeski P, Pavlovskaja E, Kapitaniak T, Perlikowski P. The application of inerter in  
425 tuned mass absorber. *International Journal of Non-Linear Mechanics* 2015;70:20–9.
- 426 [27] Chen MZQ, Hu Y, Huang L, Chen G. Influence of inerter on natural frequencies of  
427 vibration systems. *Journal of Sound and Vibration* 2014;333(7):1874–87.
- 428 [28] Chen MZQ, Smith MC. Electrical and mechanical passive network synthesis. In *Recent*  
429 *Advances in Learning and Control*, New York: Springer-Verlag; 2008;371:35–50.
- 430 [29] Chen MZQ, Smith MC. A note on tests for positive-real functions. *IEEE Trans Autom*  
431 *Control* 2009;54(2):390–3.
- 432 [30] Chen MZQ. A note on PIN polynomials and PRIN rational functions. *IEEE Transactions*  
433 *on Circuits and Systems II: Express Briefs* 2008;55(5):462–3.
- 434 [31] Chen MZQ, Smith MC. Restricted complexity network realizations for passive mechan-  
435 ical control. *IEEE Trans Autom Control* 2009;54(10):2290–301.
- 436 [32] Wang K, Chen MZQ. Generalized series-parallel RLC synthesis without minimization  
437 for biquadratic impedances. *IEEE Trans on Circuits and Systems II: Express Briefs*  
438 2012;59(11):766–70.
- 439 [33] Chen MZQ, Wang K, Zou Y, Lam J. Realization of a special class of admittances  
440 with one damper and one inerter for mechanical control. *IEEE Trans Autom Control*  
441 2013;58(7):1841–46.
- 442 [34] Chen MZQ, Wang K, Shu Z, Li C. Realizations of a special class of admittances with  
443 strictly lower complexity than canonical forms. *IEEE Trans Circuits and Systems–I:*  
444 *Regular Papers* 2013;60(9):2465–73.
- 445 [35] Chen MZQ, Wang K, Yin M, Li C, Zuo Z, Chen G. Synthesis of  $n$ -port resistive net-  
446 works containing  $2n$  terminals. *International Journal of Circuit Theory & Applications*  
447 2015;43(4):427–437.
- 448 [36] Shearer JL, Murphy AT, Richardson HH. *Introduction to System Dynamics*. Reading,  
449 MA: Addison-Wesley; 1967.
- 450 [37] Hixson EL. Mechanical impedance. In: Harris CM, editor. *Shock and Vibration Hand-*  
451 *book*, 3rd ed, New York: McGraw-Hill; 1988, ch. 10.

- 452 [38] Wang K, Chen MZQ, Hu Y. Synthesis of biquadratic impedances with at most four  
453 passive elements. *Journal of the Franklin Institute* 2014;351(3):1251–67.
- 454 [39] Asami T, Wakasono T, Kameoka K, Hasegawa M, Sekiguchi H. Optimum design of  
455 dynamic absorbers for a system subjected to random excitation. *JSME International*  
456 *Journal Series III* 1991;34(2):218–26.
- 457 [40] Cheung YL, Wong WO.  $H_2$  optimization of a non-traditional dynamic vibration abosor-  
458 ber for vibration control of structures under random force excitation. *Journal of Sound*  
459 *and Vibration* 2011;330(6):1039–44.
- 460 [41] Doyle JC, Francis BA, Tannenbaum AR. *Feedback Control Theory*. Oxford: Maxwell  
461 Macmillan Int; 1992.
- 462 [42] Inman DJ. *Engineering Vibration*. 3rd ed. NJ: Prentice-Hall; 2008.
- 463 [43] Cheung YL, Wong WO. H-infinity optimization of a variant design of the dynamic vibra-  
464 tion absorber—Revisited and new results. *Journal of Sound and Vibration* 2011;330:3901–  
465 12.