

1 Daytime Passive Radiative Cooling by Ultra
2 Emissive Bio-inspired Polymeric Surface

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15

16 **Abstract**

17 Saharan silver ants can maintain their body temperature below ambient air due to unique triangular
18 shaped hairs that enhance solar reflection and thermal emission through a transparent window that
19 lies in the atmosphere. Applying this thermoregulatory prismatic structure to polydimethylsiloxane
20 (PDMS), highly emissive in the 8-13 μm spectrum, we present a geometrically modified polymer-
21 based daytime passive radiative cooler. The selective thermal emitter was fabricated based on the
22 optimized prismatic structure from Finite Difference Time Domain (FDTD) simulations. The
23 average emissivity within the 8-13 μm spectrum was enhanced to 0.98 by the gradient refractive
24 index effect, while the average solar reflectivity in the visible and near-infrared spectrum was
25 measured to be 0.95. The net radiative cooling power is estimated to reach 144 W/m^2 , exceeding
26 records of previously reported radiative coolers. Last, in Hong Kong's hot and humid climate, a
27 field test successfully demonstrated cooling by 6.2 $^{\circ}\text{C}$ below the temperature of ambient air
28 corresponding to a net cooling power of 19.7 W/m^2 in a non-vacuum setup during the peak daytime
29 with shading. This is the largest temperature reduction observed in a tropical region for daytime
30 passive radiative cooling. Our work presents an alternative method to enhance passive thermal
31 emission and may facilitate its world wide application in eco-friendly space cooling.

32

33 *Keywords: Gradient refractive index, Mie-scattering, Radiative cooling, Saharan silver ant,*
34 *Selective emission, Thermal radiation*

35

36 **1. Introduction**

37 Daytime passive radiative cooling proposes a new eco-friendly strategy to solve the high energy
38 demand for building space cooling. Conventional and widely used cooling methods require

39 external power sources or devices where high energy consumption is inevitable due to the
40 necessities of everyday life. In contrast to the cooling methods utilizing resources and energy,
41 passive radiative cooling solely focuses on the natural cooling strategies of near perfect solar
42 energy reflection which lies in the visible (VIS) and near-infrared (NIR) spectrum, minimizing
43 thermal absorption of incident solar irradiation. Furthermore, in the mid-infrared (MIR) spectrum
44 (8-13 μm), the strong selective thermal radiation occurs from an object efficiently delivering
45 thermal energy to the cold heat sink of outer space ($\sim 3\text{ K}$) through the atmospheric transparency
46 window where radiation absorption is very low in the 8-13 μm spectrum. Nocturnal passive
47 radiative cooling has been successfully conducted [1–10]; however, high cooling demands occur
48 at peak hours during the daytime.

49

50 Recently, scientists and engineers have demonstrated a sky facing surface able to sustain its
51 temperature below the temperature of ambient during daytime with a passive selective radiation
52 strategy. A typical configuration of a daytime passive radiative cooler can be elaborated as two
53 layered structures, including the top emissive layer and bottom reflective layer. Various materials
54 and geometrical modifications in the top emissive layer were investigated to produce a cooling
55 effect during daytime, such as photonic, plasmonic, biomimetic materials, etc. A photonic radiative
56 cooler with a thermal emitter composed of seven repetitive layers of hafnium oxide (HfO_2) and
57 silicon dioxide (SiO_2) and a solar reflector of silver (Ag), demonstrated a radiated power of 40
58 W/m^2 and a daytime cooling performance of nearly $5\text{ }^\circ\text{C}$ reduction below the temperature of
59 ambient air [11]. This nanophotonic device which showed promising cooling performance in a dry
60 region was tested in the sub-tropical climate of Hong Kong [12, 13]. A temperature reduction of
61 $6\text{-}7\text{ }^\circ\text{C}$ was achieved under a clear night sky, but daytime cooling was not achieved in the humid

62 climate since the mid-infrared thermal emission is weakened by the huge amount of atmospheric
63 water vapor. The nano-phonic approach requires precise and accurate fabrication, thus, the
64 simplified radiative cooler was developed by deposition of silicon nitride (Si_3N_4), and amorphous
65 silicon (Si) on the top of a solar reflector of aluminum (Al) and in a vacuum condition, a strong
66 daytime radiative cooling was achieved by 42 °C reduction below the ambient in the dry climate
67 of California, USA [14]. A polymer based simplified passive cooler was proposed, designed with
68 a polydimethylsiloxane (PDMS) polymer on top of the silver (Ag) sputtered glass (SiO_2) wafer
69 achieving a daytime cooling of 8.2 °C below the ambient relevant to a high radiative power of
70 127 W/m^2 in low humidity California [15]. Photonic crystal structures theoretically and
71 experimentally demonstrated selective thermal emission by surface plasmonic resonance [16–29].
72 Different types of plasmonic radiative coolers were proposed including micropillar patterns on Al
73 plate [30] and Ag deposited micro-patterned silicon (Si) wafers [31] demonstrating a numerically
74 analyzed net cooling power of 100 W/m^2 . Generally, literatures report on lab scale radiative coolers
75 with limited practical applications. Comprehensive reviews on a scaled-up glass-polymer hybrid
76 radiative cooler by reel-to-reel processing has been carried out [32, 33]. Motivated by a polymeric
77 radiative cooler with silicon carbide (SiC) and dispersed SiO_2 micro-particles [34], a scalable SiO_2
78 micro-particle doped polymeric radiative cooler with the optimized particle size enhancing
79 selective thermal emission in mid-infrared (MIR) emission was manufactured, recording a
80 radiative power of 93 W/m^2 [33]. Daytime passive radiative coolers can be directly attached to
81 fabrics [35, 36] and electronics [37, 38] for efficient cooling, or they can be integrated in a
82 refrigeration cycle of HVAC systems in buildings as a condenser which saves energy consumed
83 in space cooling due to its cooling performance without energy input [39-41].

84

85 Both organic and inorganic materials with strong emission property in the MIR wavelength
86 spectrum (8-13 μm) were used as the base thermal emission substrate for passive radiative cooling
87 [11–34]. In this study, for the first time, photonic approach on polymer was utilized to design a
88 passive radiative cooler where geometry modification in polymer has been seldom considered to
89 further enhance emission properties due to the challenge of controlling defects. However, polymer
90 is a very attractive material due to its low cost, compatibility with various patterns, offering the
91 required optical, electronic, and mechanical properties required [34, 42]. This work has been
92 triggered by the Saharan silver ant shown in Fig. 1a that possesses two different passive thermal
93 regulatory effects and successfully survives in extremely hot deserts. The Saharan silver ant has
94 unique shaped hairs, in the form of triangular prisms, providing two thermoregulatory effects as
95 shown in Fig. 1b that allows the ant to keep cool under the extremely strong sunlight of the Saharan
96 desert [43–45]. The triangular shaped hair induces enhancement in total internal reflection within
97 the solar spectrum (0.25 – 2.5 μm) resulting in high reflection of incident solar irradiance. The
98 enhancement is realized by Mie scattering which takes place in situations where the size of the
99 object is comparable to the wavelength of incident light, and for the ant, the scattering effect occurs
100 at $\sim 2 \mu\text{m}$ [46]. Furthermore, enhanced MIR emissivity of the hair facilitates heat dissipation of
101 thermal energy accumulated in the ant's body which can easily escape through the transparent
102 window. Shi et al. reported that due to its strong enhancement in solar reflection, the hair structure
103 can be applied to develop a radiative cooler with a high reflective surface which does not require
104 a metal reflector. However, the triangular prismatic structures were only able to achieve solar
105 reflection of $\sim 60\%$ which is far from the solar reflection requirement for radiative coolers (i.e. $>$
106 95%) [44], thus, the application approach was newly established to enhance MIR thermal radiation
107 of the daytime radiative cooling by utilizing gradual refractive index change effective in 8-13 μm .

108

109 In this study, we propose a selective thermal emission device with application of the unique
110 triangular prism shape of the Sahara silver ants' hair to perform enhanced radiative cooling
111 performance. First, we investigated improvement in MIR thermal emission utilizing the micro-
112 scale triangular prism structure compared to a conventional flat, multi-layered film and hair
113 resembling a circular tube-structured cooler. The following study was conducted on the triangular
114 structure for optimization to achieve the strongest thermal emission by controlling parameters such
115 as the geometrical configurations and size. After computational optimization of the triangular
116 structure, a passive radiative cooler employing the triangular prism array was then manufactured
117 by a microfabrication process. To verify the result of the simulation study, the optical characteristic
118 of the fabricated cooler was measured and a comparative study with the uniformly layered cooler
119 was conducted. Based on the measured optical properties, its cooling performance was analyzed
120 numerically with energy balance equations. Lastly, to test its practicality under the humid weather
121 condition of Hong Kong with a low transparent atmospheric window, field investigations were
122 carried out and the cooling performance was observed. This study indicates geometrical
123 modification as a potential solution to improve selective thermal emission besides the selection of
124 various materials, which can significantly improve the passive radiative cooling effect by scaling
125 down, reducing materials required for manufacturing but enhancing the cooling power. This
126 indicates that the biomimetic radiative cooler could be an attractive non-energy-consuming
127 solution to reduce air conditioning costs for buildings in the near future.

128

129 **2. Material and methods**

130 **2.1. Design and optimization method**

131 In this study, we propose the bio-inspired passive daytime radiative cooler with a design of a top
132 MIR emissive layer of PDMS and SiO₂ that strongly radiates in the 8-13 μm spectrum and a bottom
133 solar reflective layer composed of Ag which can provide solar reflection above 95 %. Optimization
134 was mainly focused on achieving the highest emissivity within the targeted MIR range (8-13 μm)
135 due to the major contribution in net radiative cooling performance. PDMS and SiO₂ are already
136 well known for being selectively emissive in 8-13 μm wavelength range due to a high value of
137 imaginary permittivity [47]. Furthermore, the proposed cooler possesses an advantage of easy
138 fabrication due to widely available SiO₂ substrate wafer (400-500 μm) and flexible material
139 property of PDMS which can be easily shape to desired geometries on the surface by a nano-
140 imprinting process. The great freedom to shape PDMS allows us to study the improvement of
141 optical properties especially emissivity in different configurations. The geometrical structure on
142 the surface of the PDMS emitter was optimized by utilizing FDTD simulations considering
143 different geometries and sizes. After optimization, a comparison study regarding MIR emission
144 between our optimized bio-inspired cooler and conventional flat surface cooler was performed.
145 Prior to the fabrication and field experiment, FDTD optical computational analysis which directly
146 solves Maxwell equations was utilized to achieve the highest net cooling power by optimizing
147 geometrical structures of the PDMS emitter surface and the overall design of the cooler.

148

149 **2.2. Fabrication**

150 For the microfabrication of the optimized bio-inspired daytime radiative cooler, a mold with
151 triangular prism patterns needs to be fabricated first which can later be used to print patterns on
152 the PDMS layer. A 525 μm thick P-type 4” silicon wafer was used as the substrate for the triangular
153 prism array mold. The silicon wafer first underwent pre-photolithography cleaning in Piranha

154 solution, mixture of 10:1 (v/v) $\text{H}_2\text{SO}_4:\text{H}_2\text{O}_2$, at 120 °C for 10 mins to remove organic residues and
155 gross contaminants. Native oxide layer on the silicon wafer surface was removed by hydrofluoric
156 acid solution, mixture of 1:50 (v/v) $\text{HF}:\text{H}_2\text{O}$, for 1 min. After the cleaning process, a thin silicone
157 oxide layer of 60 nm was produced on the silicon wafer surface by thermal oxidation using a
158 diffusion furnace at a temperature between 800 - 1200 °C. The purpose of the silicon oxide layer
159 is to protect the unexposed silicon area during an etching process. Triangular prism arrays were
160 photo-lithographically patterned on a 1- μm -thick layer of photoresist (PR), HPR 504, on top of the
161 silicon dioxide layer. The pattern consists of repetitive rectangular lines with a uniform width and
162 a gap of 1.5 μm between each line. Patterns for different line widths were considered in fabrication
163 varying from 8-13 μm to study the cooling performance of different triangular sizes. The silicon
164 dioxide layer was etched using an advanced oxide etching (AOE) process that etches only the
165 exposed photoresist layer until the silicon surface is exposed. The photoresist layer was solely
166 required for leaving the pattern on the silicon dioxide layer and therefore, after etching of the
167 silicon dioxide layer, it was completely stripped by O_2 plasma treatment. Before the silicon etching
168 process started, the silicon wafer was treated with hydrofluoric acid again for complete removal
169 of native oxide on the exposed silicon surface which might hinder the process of silicon etching.
170 The silicon wafer was then immersed into a tetramethylammonium hydroxide (TMAH) solution
171 (25 wt. %), a silicon etchant, at temperature of 90 °C and only the unprotected silicon area was
172 etched. TMAH etches silicon leaving V-grooves and the angle between the sidewalls and the (100)
173 plane is 54.7 °. Anisotropic etching property of TMAH achieves triangular prism arrays on the
174 silicon wafer. Typical etch rate of TMAH for the (100) plane was 1.0 $\mu\text{m}/\text{min}$ and for the (110)
175 plane was 1.4 $\mu\text{m}/\text{min}$ at 90 °C [48]. The etching time was carefully controlled based on the widely
176 known etching rate of silicon. Silicon dioxide layer was perfectly removed by buffered oxide etch

177 (BOE) solution, mixture of 6:1 (v/v) $\text{NH}_4\text{F}:\text{HF}$, and the mold for the triangular prism array
178 structure was completed. Totally, three molds for different triangular size parameter of 8, 9, and
179 $10\ \mu\text{m}$ were fabricated. The patterned surface of the silicon mold was sputtered with 320 nm of
180 chromium which acts as a PDMS anti-adhesion layer. PDMS and its curing agent were mixed
181 together in the mass ratio of 10:1 then spin-coated with $20\ \mu\text{m}$ thickness on a $525\ \mu\text{m}$ thick $4\ \text{cm}$
182 $\times 4\ \text{cm}$ glass substrate with 160 nm thick silver sputtered at the back. On top of the spin-coated
183 PDMS, the silicon mold facing patterned surface toward PDMS was pressed gently and cured in
184 an oven at $80\ ^\circ\text{C}$ for 4 hrs. After the curing process finished, the patterned silicon mold was gently
185 removed, achieving a three layered passive radiative cooler sample consisting of a triangular prism
186 array structured PDMS, glass and silver at the bottom. **The fabrication process is elaborated more**
187 **in Fig. S1 in the Supplementary Materials.**

188

189 **3. Theory/calculation**

190 To carry out the cooling performance analysis numerically for the fabricated bio-inspired cooler
191 such as the net cooling power and achieved temperature reduction below the temperature of
192 ambient air, the energy balance state of the daytime passive radiative cooling was carefully
193 investigated. The net radiative cooling power, P_{cool} , is shown below [11]:

194

$$195 \quad P_{cool} = P_{rad}(T_{prc}) - P_{atm}(T_{atm}) - P_{sun} - P_{con}(T_{prc}, T_{atm}), \quad (1)$$

196

197 where P_{rad} is the thermally radiated power of the cooling device [W], T_{prc} is the surface
198 temperature of the cooler [K], P_{atm} is the power absorbed by the passive radiative cooler from the
199 ambient atmosphere [W], T_{atm} is the ambient atmosphere temperature [K], P_{sun} is the absorbed

200 power by the surface of the cooler from the incident solar irradiance [W], and P_{con} is the convective
 201 and conductive power gained by the cooler surface [W]. The radiative thermal energy emitted from
 202 the cooler at the cooler temperature, T_{prc} , is given by:

$$204 \quad P_{rad}(T_{prc}) = A_{prc} \int d\Omega \cos\theta \int_0^\infty d\lambda I_{BB}(T_{prc}, \lambda) \varepsilon(\lambda), \quad (2)$$

203
 206 where A_{prc} is the area of the sky facing cooler [m²]. $I_{BB}(T, \lambda) = \frac{2hc^2}{\lambda^5} \frac{1}{e^{hc/(\lambda k_B T)} - 1}$ is the
 207 spectral distribution of the thermal energy radiated by a black-body [W/(m³sr)] at any temperature
 208 T and h is Planck's constant of 6.63×10^{-34} J·s, c is the universal physical constant for speed of light
 209 which is 3.00×10^8 m/s, λ is wavelength [m], and k_B is denoted as the Boltzmann constant of
 210 1.38×10^{-23} J/K, $\varepsilon(\lambda)$ refers to the emissivity spectra (0.25- 25 μ m) of the radiative cooling devices
 211 and the measured emission spectrum of the coolers used. $P_{atm}(T_{atm})$ is the absorbed power by the
 212 cooler surface from the thermal radiation emitted from the surrounding atmosphere at the
 213 temperature of T_{atm} and can be defined as:

$$215 \quad P_{atm}(T_{atm}) = A_{prc} \int d\Omega \cos\theta \int_0^\infty d\lambda I_{BB}(T_{atm}, \lambda) \varepsilon(\lambda) \varepsilon_{atm}(\lambda), \quad (3)$$

214
 217 where $\varepsilon_{atm}(\lambda)$ is the emission spectrum (0.25-25 μ m) of the atmosphere [49]. Validation of the
 218 numerical model was carried out by comparing the numerically estimated cooling power with
 219 previous studies [11–15, 33] by using the widely known atmospheric transmittance data measured
 220 in Mauna Kea located in Hawaii, U.S.A. with conditions of 1.5 air mass and precipitable water
 221 vapor of 1.0 mm [50]. Thermal energy is emitted from the surrounding atmosphere and absorbed
 222 by the radiative cooler with the ratio of its emissivity degrading net cooling performance. The law

223 of Kirchhoff's radiation clearly states that the material's absorptivity and emissivity can be
224 considered equal because under a thermodynamic equilibrium state, the fraction of an emissive
225 power from a perfect black-body shows it to be equivalent to its ratio of the incoming power to the
226 absorbed power of the object [2–10].

227

$$228 \quad P_{sun} = A_{prc} \int_0^{\infty} d\lambda I_{AM1.5}(\lambda) \varepsilon(\lambda), \quad (4)$$

229

230 is the absorbed power by the radiative cooler from incoming solar irradiance. $I_{AM1.5}$ is the **AM**
231 **1.5 G** spectrum which can be defined by the solar intensity of densely populated regions which are
232 located at moderate altitude [51] and used widely in the photovoltaic industry [52]. P_{con} is the
233 parasitic heat delivered to the cooler from the surrounding atmosphere by conduction and
234 convection and can be defined as:

235

$$236 \quad P_{con}(T_{prc}, T_{atm}) = A_{prc} h_{con} (T_{atm} - T_{prc}), \quad (5)$$

237

238 where h_{con} is the heat transfer coefficient for the heat transfer between the cooler and the
239 surrounding atmosphere due to conduction and convection [W/(m²K)].

240

241 **4. Results and Discussion**

242 **4.1. FDTD optimizations**

243 The emission spectra were investigated for three different geometrical configurations of the
244 PDMS thermal emitter with various characteristic lengths of the proposed geometries. These
245 configurations are conventional uniform flat surfaces, ordinary hair resembling periodic circular

246 arrays, and lastly, Saharan silver ant hair inspired periodic triangular arrays. The optical property
247 comparison in different geometrical structures is to show that the triangular prism shape adopted
248 surface can enhance MIR emissivity of the conventional flat emitter. The PDMS emitter with
249 circular arrays which represents commonly recognized hair was also studied to further investigate
250 the uniqueness of the triangular hair structure compared to the ordinary hair structure. Referring
251 to Fig. 1c, characteristic lengths (a) of the geometries were selected as the thickness of the uniform
252 flat surface, the diameter of the circle, and the bottom length of the triangle. The triangular
253 structure was set to isosceles triangle with equal angle of 54.7° because the angle is limited to that
254 for the wet etching process on a silicon wafer. Knowing that Mie scattering occurs when incoming
255 wavelength and object share similar size, characteristic lengths of the proposed structure were
256 selected within the range of 2-15 μm to investigate MIR emissivity enhancement of the emitter
257 caused by the scattering effect. Fig. 1c shows the averaged MIR emissivity of three different
258 geometries along the characteristic lengths. This plot represents the PDMS emitter with triangular
259 arrays on the surface, always achieving a stronger thermal emission property than the other two
260 geometrical configurations. Furthermore, when the size parameter of the triangle was between 8-
261 $13\mu\text{m}$, the simulated average emissivity within the atmospheric transparent window showed a
262 higher value than when the size was outside this range. This is mainly due to the triangular prism
263 structure with a size between 8-13 μm that can provide a gradual change in the refractive index in
264 8-13 μm between air and PDMS. Impedance mismatch occurring between air and PDMS can be
265 reduced, enhancing the overall emissivity of PDMS. Different size of triangular prism PDMS
266 gradient index layer within the size parameter of 8-13 μm shows varying influence on emissivity
267 within this bandwidth due to the difference in resonance points. The size parameter of 8 μm which
268 is four times larger than the original hair showed the strongest improvement in thermal emission

269 within 8-13 μm , resulting in an average emissivity value of 0.98. Fig. 1d shows the emission
270 spectra for three different configurations sharing the same characteristic length of 8 μm in the
271 wavelength range of 8-13 μm . At the same characteristic length, the triangular PDMS emitter
272 clearly shows the highest MIR emissivity over the other two emitters. Referring to the FDTD
273 simulation study, triangular structure is the best for enhancing thermal emission within MIR and
274 the triangle with optimized characteristic length of 8 μm induces the most Mie scattering within
275 8-13 μm , resulting in the highest average emissivity of 0.98 within 8-13 μm .

276

277 **4.2. Optical measurement and numerical analysis**

278 Based on the simulation study, a PDMS-SiO₂-Ag three layered cooler, in which the PDMS
279 thermal emitter (4-inch wafer) shown in Fig. 2a was of an optimized triangular geometry with a
280 characteristic length of 8 μm , was micro-fabricated. Silicon molds with triangular arrays shown in
281 Fig. 2b were used to print patterns on the PDMS surface. Fig. 2c shows the enlarged cross-section
282 of the silicon mold. Validation of the simulation results was followed by investigating PDMS
283 emitters with different characteristic lengths of the triangle, 8 μm , 9 μm and 10 μm , together with
284 a uniform 100 μm thick PDMS emitter. To investigate the overall cooling performance
285 improvement of the bio-inspired cooler compared to the previously proposed radiative coolers
286 [15], the uniform 100 μm thick PDMS emitter was chosen rather than a uniform PDMS emitter
287 with a similar characteristic length of triangular patterned PDMS emitter. To compare and
288 understand the optical properties of the samples, the emissivity of the samples over the ultraviolet
289 (UV) to IR wavelength ranges shown in Fig. 3a were measured by an UV/VIS/NIR spectrometer
290 and Fourier transform infrared spectroscopy (FTIR). Fig. 3b shows the emissivity spectrum in the
291 UV to NIR region of the coolers with PDMS in uniform and triangular structures with size

292 parameters at 8, 9, and 10 μm . Solar radiation absorbed by the coolers can be numerically
293 calculated by the multiplication of the measured absorptivity of coolers within UV-NIR and AM
294 1.5 G solar irradiance at unit wavelength. All four different samples of radiative coolers showed
295 low solar absorption power density about 20-25 W/m^2 that mainly absorbed within the ultraviolet
296 spectrum. The triangular prism structure did not show any improvement in solar reflection but
297 mainly enhanced the thermal radiation of the passive radiative cooler. Fig. 3c shows the emissivity
298 spectra of the four different coolers in the wavelength range of 8–13 μm . The average emissivity
299 for the uniform PDMS cooler was 0.92 and the triangular PDMS cooler with a size parameter of
300 8 μm showed 0.98, an enhancement of 6.5%. The triangular PDMS cooler with a size parameter
301 of 10 μm showed the lowest emissivity (0.93) among the patterned PDMS coolers, indicating the
302 triangular structure to have better emission performance within the 8-13 μm spectrum range than
303 the uniform PDMS cooler. Lastly, a comparison solar reflectivity and MIR emission spectrum of
304 the previously reported daytime passive radiative coolers [11, 13-15, 53, 54] with the triangular
305 patterned PDMS cooler was conducted to understand the performance improvement. Fig. 4a shows
306 that the bio-inspired triangular PDMS cooler is not really outstanding in solar reflectance among
307 all the radiative coolers previously studied. Due to the wide usage of silver as a reflector in the
308 daytime passive radiative coolers, most coolers including the bio-inspired triangular PDMS cooler
309 (i.e. this work) show high solar reflection (i.e. at least 97%). The highest solar reflection was
310 achieved by the PTFE-Ag cooler which can reflect 99% of incoming solar irradiance, absorbing
311 only $\sim 10 \text{ W}/\text{m}^2$, which is $15 \text{ W}/\text{m}^2$ less than the bio-inspired triangular PDMS cooler (i.e. this
312 study) [53]. However, Fig. 4b shows that the bio-inspired triangular PDMS cooler shows very
313 clear difference in MIR emissivity by having almost unity spectrum in the 8-13 μm wavelength.
314 This obvious distinction shows the developed radiative cooler has almost achieved the maximum

315 **emission performance for the daytime passive radiative cooling technique.** The prismatic structure
316 obviously showed superiority in MIR emissivity compared to the uniform surface and with the
317 measured optical characteristics, the cooling performance improvement can be investigated
318 numerically.

319

320 To validate the result analyzed by the energy balance equation, the cooling performance of a
321 fabricated uniform 100 μm thick PDMS silica-mirror was analyzed based on its measured optical
322 properties and compared with the previously reported value [15]. When there are no parasitic heat
323 gains from the ambient atmosphere (i.e. $h_{con} = 0 \text{ W}/(\text{m}^2\text{K})$), Fig. 5a shows that the fabricated
324 uniform 100 μm PDMS cooler can passively provide a net cooling power of $127 \text{ W}/\text{m}^2$ at the
325 temperature of the surrounding atmosphere, T_{atm} , of $27 \text{ }^\circ\text{C}$ that perfectly matches with the
326 previously reported net cooling power of the uniform 100 μm thick PDMS silica-mirror [15]. The
327 patterned PDMS cooler can provide the maximum net cooling power of $144 \text{ W}/\text{m}^2$ at the ambient
328 air temperature of $27 \text{ }^\circ\text{C}$ which corresponds to an enhancement of 13.4 % compared to uniform
329 PDMS cooler. The ideal passive radiative emitter, perfect solar reflection in the spectrum of $0.3 -$
330 $2.5 \mu\text{m}$ and perfect emission in the spectrum of $8 - 13 \mu\text{m}$, was estimated to provide a maximum
331 net radiative cooling power of $132 \text{ W}/\text{m}^2$ at the ambient air temperature of $27 \text{ }^\circ\text{C}$ [55]. However,
332 the cooling performance can be further enhanced by exploiting spectrum regions outside the
333 atmospheric transparent window, $2.5 - 8 \mu\text{m}$ and $13 - 25 \mu\text{m}$. Utilizing this broad spectrum region,
334 $2.5 - 25 \mu\text{m}$, an ideal radiative cooler with perfect solar reflection in the spectrum of $0.3 - 2.5 \mu\text{m}$
335 and perfect emission in the spectrum of $2.5 - 25 \mu\text{m}$ can produce $207 \text{ W}/\text{m}^2$ maximum cooling
336 power at the ambient of $27 \text{ }^\circ\text{C}$ [55]. The total thermal radiation absorbed from the atmosphere to
337 the cooler is increased but the outgoing thermal emission from the cooler exceeds the incoming

338 radiation, thereby enhancing the net cooling performance. The triangular structure greatly
339 enhances emissivity within 8 - 13 μm and outside the main atmospheric transparent window,
340 ultimately exceeding the cooling power of the narrow-band ideal radiative cooler. Fig. 5b presents
341 an estimated cooling power of the cooler at a condition where the ambient, T_{atm} , is 27 °C and a
342 parasitic heat coefficient of $h_{con} = 10 \text{ W}/(\text{m}^2\text{K})$ [15]. In Fig. 5b, temperature of the cooler is
343 reduced below the ambient air temperature by 8.7 °C for the uniform 100 μm PDMS cooler. The
344 patterned PDMS cooler can decrease the surface temperature by a maximum of 9.8 °C which
345 shows the triangular structure greatly enhances the cooling performance compared to the uniform
346 PDMS cooler.

347

348 **4.3. Field investigation**

349 To validate the numerically estimated cooling performance and study the actual cooling behavior
350 in non-ideal atmospheric conditions, 24-hour field investigations were conducted under the sky
351 condition of Hong Kong. The radiative cooling performance investigation was performed on the
352 building roof top where the cooler can perfectly face the sky. In the experiment setup, minimizing
353 the parasitic heat gains to the coolers was the top priority in order to maximize the net cooling
354 power. The experiment site was perfectly covered by a calcium-magnesium silicate thermal
355 insulation sheet to minimize the conductive heat generated from the concrete surface heated from
356 the sunlight. Petri dishes with three acrylic legs which can minimize the contact surface with the
357 ground were placed on top of the sheet. The fabricated coolers were placed in the Petri dishes to
358 limit the thermal contact with the ambient air. Additional acrylic supporting structures were also
359 fabricated to minimize the contact with the Petri dishes. Finally, to minimize the convective heat
360 transfer by wind, the Petri dishes were covered by a very thin polyethylene film which is

361 transparent within the wavelength ranges of solar radiation and passive radiative cooling. A
362 detailed schematic design of the experimental setup can be found in Fig. 6a. Fig. 6b shows the
363 experimental setup on the rooftop of a building. A data acquisition device, National Instruments
364 NI 9213, was used to record (frequency: 1 s, accuracy: ± 0.02 °C) the temperatures of coolers and
365 ambient air measured by thermocouples. Thermocouples used in the field investigations were
366 calibrated to eliminate measurement error and directly attached to the back side of the cooler
367 having the silver layer. The experiment was conducted during clear days of June and September.

368

369 Totally, 2 different samples were prepared to carry out the comparative study. These consisted
370 of a uniform 100 μm PDMS-SiO₂-Ag cooler and an 8 μm triangular patterned PDMS-SiO₂-Ag
371 cooler that showed the highest net cooling power among all the patterned coolers with different
372 sizes of triangle. Fig. 7a presents the four different temperature profiles of the 8 μm triangular and
373 the 100 μm thick uniform PDMS-SiO₂-Ag coolers with and without solar shade for a 48 hr cycle
374 measurement in Hong Kong (Date: Sep-30-2018 to Sep-31-2018). The average relative humidity
375 of these two days was 87 % and the sky was very clear with 0-1 oktas. Under the humid Hong
376 Kong weather, no matter shaded or unshaded, during the night time, both the triangular patterned
377 and the uniform cooler showed cooling effects with a temperature reduction of 6.1 °C and 5.2 °C
378 below the ambient air temperature, respectively. The triangular patterned PDMS cooler showed a
379 lower surface temperature by average 1 °C than the uniform PDMS cooler. The uniform PDMS
380 cooler was reported to have a temperature decrease of 8.4 °C during night time of Pasadena,
381 California [15]. The lower value of the measured temperature reduction below the ambient during
382 the night time is due to the high relative humidity in Hong Kong. The transparency of the
383 atmospheric window that lies in 8-13 μm spectrum is closely affected by the vapor concentration

384 in the atmosphere that results in low transparency, i.e., high absorption of atmosphere is due to the
385 high relative humidity and this limits the escape of the thermal radiation to the cold universe. The
386 MIR absorption of atmosphere enhanced by three-folds as the precipitable water vapor increases
387 from 1.5 cm to 6 cm [3], showing the limitation of the passive radiative cooling in humid weather.
388 Conditions of low humidity and cloudless sky can result in strong transmittance in the MIR
389 spectrum facilitating the radiative cooler to produce the ideal cooling performance. However, the
390 field investigation showed that the triangular pattern could enhance the radiative cooling effect,
391 reducing the temperature nearly 1 °C more than the uniform PDMS cooler. Fig. 7b (i.e. a zoom-in
392 figure) shows the four different temperature profiles of the 8 μm triangular and the 100 μm thick
393 uniform PDMS-SiO₂-Ag coolers under shaded and unshaded conditions during the peak daytime
394 of Hong Kong (Date: Sep-31-2018). During the daytime, even under the clear sky, 0-1 oktas, it is
395 clear that both coolers without shading were unable to produce a cooling effect. At the peak
396 daytime, both the triangular PDMS cooler and the uniform PDMS cooler without shading was
397 measured with a temperature higher than the ambient air temperature by 5.1 °C and 6.0 °C,
398 respectively. Due to the characteristics of the tropical climate, both coolers, without shading, were
399 unable to deliver cooling performance under the peak solar irradiation as incident solar energy
400 exceeded the radiated thermal energy from the coolers to the atmospheric transparency window.
401 However, even in this poor weather condition for radiative cooling, the triangular structured cooler
402 could maintain a temperature 1 °C below the temperature of the uniform cooler. In order to provide
403 cooling during the peak of Hong Kong's daytime, solar shades, fabricated by aluminum sheet,
404 were installed to shade the coolers during daytime to minimize the incident solar energy delivered
405 to the coolers. With the solar shades partially blocking the incoming sunlight, at the peak daytime,
406 daytime cooling was achieved with the maximum temperature reduction (average temperature

407 difference between 12:00 – 13:00 (Date: 31-Sep-2018)) of 6.2 °C and 5.1 °C below the ambient
408 for the triangular and uniform, respectively.

409

410 Net radiative cooling power measurement of both a uniform 100 μm PDMS-SiO₂-Ag cooler and
411 an 8 μm triangular patterned PDMS-SiO₂-Ag cooler was conducted in a humid climate condition
412 of Hong Kong. Silicon heating mats with a heating capacity of 20 W were attached at the back of
413 both coolers. The cooling power produced by the radiative cooler will be equal to the heating
414 power from the heater when its temperature remains the same as the ambient air temperature. Thus,
415 electrical current flow delivered to the cooling power measurement system was controlled by a
416 feedback loop to maintain the temperature of the cooler equal to the ambient. The net cooling
417 power of these two coolers was estimated by summing up electrical energy consumed by the heater
418 pad in an hourly rate. Fig. 7c presents the collected data for temperature and net cooling power of
419 the uniform 100 μm PDMS-SiO₂-Ag cooler and the 8 μm triangular patterned PDMS-SiO₂-Ag
420 cooler. The peak ambient air temperature during the daytime was measured near 33 °C at 2 pm
421 (08-June-2019), when 19.7 W/m² cooling power was recorded for the triangular patterned PDMS
422 cooler, while a net cooling power of 17.9 W/m² was obtained from the uniform PDMS cooler.
423 During nighttime operation, the lowest temperature of ambient air was measured at 19 °C (5 am,
424 08-June-2019) and the triangular patterned PDMS cooler produced 14.3 W/m² cooling power,
425 while 13.3 W/m² cooling power was recorded for the uniform PDMS cooler. Lower temperature
426 of ambient at night weakened the thermal radiation emitted from both coolers resulting in lower
427 nocturnal cooling power compared to the daytime cooling. In total, both patterned and non-
428 patterned radiative coolers showed weakened cooling performance compared to the outcome

429 achieved in the clear and dry climatic conditions. Nevertheless, the triangular patterned PDMS
430 cooler showed a 10% enhanced net cooling power than that of the uniform PDMS cooler.

431
432 Table 1 summarizes the recent achievements in daytime passive radiative cooling regarding their
433 structural designs, optical properties including solar reflectance and MIR emissivity, and cooling
434 performance. Uniform flat PDMS radiative cooler demonstrated experimentally the highest
435 daytime cooling performance by generating 127 W/m^2 [15]. Optimized $\text{TiO}_2\text{-SiO}_2$ alternating
436 multi-layered cooler theoretically predicted a higher net cooling power of 136.6 W/m^2 while its
437 field investigation demonstrated only 14.3 W/m^2 due to the poor sky condition of sub-tropical
438 Hong Kong [13]. The bio-inspired radiative cooler in this work presents the highest cooling
439 performance numerically, showing the net cooling power of 144 W/m^2 . However, due to
440 unfavorable sky condition of low atmospheric transparency, its field investigation result shows a
441 net cooling power of 19.7 W/m^2 and temperature reduction of $6.2 \text{ }^\circ\text{C}$ under direct sunlight. While
442 a majority of the work was conducted under ideal sky conditions, it should be noted that daytime
443 passive radiative cooling is mostly needed in hot sub-tropical or tropical regions. Its cooling
444 performance deteriorates significantly in high humid weather conditions, but by minimizing
445 incoming solar irradiance, daytime cooling effect can be successfully achieved.

446

447 5. Conclusion

448 In conclusion, the bio-inspired thermal selective surface utilizing the unique triangular structure
449 of the Saharan silver ant hair was fabricated by nano-imprinting. Triangular prismatic hair
450 originally functioned to mainly enhance solar reflection, but was unable to meet the required solar
451 reflection for daytime passive cooling. Its application approach in designing a cooler was set to

452 enhance MIR emissivity by utilizing gradient refractive index change provided by the triangular
453 structure. For the first time, we have demonstrated geometrical modification on a polymeric
454 surface to enhance the emission property. The geometrical details of the triangle and the overall
455 design of the cooler were optimized through the FDTD simulation, proposing a passive radiative
456 cooler design with PDMS triangular arrays at a size characteristic of 8 μm , four times larger than
457 the original hair of the Saharan silver ant, deposited on the silver coated silica wafer. Mathematical
458 analysis on temperature reduction and cooling performance based on optical measurement were
459 conducted to validate the simulation work, proving that the bio-inspired triangular structure can
460 greatly enhance the emissivity within 8-13 μm at the maximum of 0.98, exceeding the cooling
461 performance of existing intricate nanophotonic surfaces. Theoretically, the triangular pattern can
462 enhance the net cooling power to 144 W/m^2 which is the highest value among the developed
463 daytime passive radiative coolers. Also, experimentally, the field investigation on cooling
464 performance of the triangular PDMS cooler was studied in Hong Kong to understand its
465 practicality in a humid weather climate. With the poor atmospheric transparency of Hong Kong's
466 tropical climate, an average atmospheric transmittance within 8-13 μm below 0.5, daytime cooling
467 effect could be achieved using solar shades, and the surface temperature could be sustained at a
468 maximum 6.2 $^{\circ}\text{C}$ below the ambient which corresponds to a net cooling power of 19.7 W/m^2 in a
469 non-vacuumed condition. This investigation delivers a significant meaning by demonstrating the
470 unprecedented daytime passive radiative cooling in a humid climate condition, exceeding the
471 cooling performance of leading coolers which are already available. Overall, applications of
472 triangular prism structure can greatly improve the cooling performance of a plain uniform surface
473 and can be further applied to different surfaces to optimize the desired optical properties. Further
474 modification to the triangular structure can be conducted to maximize the reflection of solar

475 irradiance to achieve a stronger passive cooling system. Our study facilitates application of passive
476 radiative cooling as a zero-energy input sustainable system to high energy consuming areas such
477 as indoor space cooling [39-41], photovoltaic industry [37, 38], personal thermal managing and
478 regulating system [35, 36].

Table 1. Summary of structural design, optical properties and cooling performance of daytime passive radiative coolers.

Authors	Structural Designs	Methodology	Solar Reflectance	MIR Emissivity	Experimental Locations & Conditions	Temperature Reduction (°C)	Cooling Power (W/m ²)
Bao et al. [56]	2 layers of TiO ₂ , SiO ₂ , and SiC nanoparticles	Numerical and Field Investigation	0.907	0.90	Shanghai, China	5	25
					Non-vacuum		
Chen et al. [14]	3 layers of 70 nm thick Si ₃ N ₄ , 700 μm thick Si, and 150 nm thick Al solar reflector at the back.	Numerical and Field Investigation	0.967	0.56	Stanford, California, USA	42	60
					Vacuum		
Jeong et al. [13]	8 alternating layers of 500 nm thick TiO ₂ and 500 nm thick SiO ₂ layers with 200 nm thick Ag solar reflector and supported by 750 μm Si layer at the back.	Numerical and Field Investigation	0.942	0.84	Hong Kong, China	7.2	14.3
					Non-vacuum		
Kecebas et al. [54]	Top 9 layers consisting of alternating TiO ₂ , SiO ₂ , and Al ₂ O ₃ each 200 nm thick. Lower 4 layers consisting of alternating TiO ₂ and SiO ₂ , each 20 nm thick. 50 nm thick Ag solar reflector at the back.	Numerical	0.960	0.69	N.A.	N.A.	103

Kecebas et al. [54]	Top 9 layers consisting of alternating TiO ₂ and SiO ₂ , each 200 nm thick. Below 4 layers consisting of alternating TiO ₂ and SiO ₂ , each 60 nm thick. 50 nm thick Ag solar reflector at the back.	Numerical	0.965	0.54	N.A.	N.A.	85.8
Kou et al. [15]	3 layers of 100 μm thick PDMS, 500 μm thick SiO ₂ , and 120 nm thick Ag solar reflector at the back	Numerical and Field Investigation	0.975	0.92	Pasadena, California, USA	8.2	127
					Non-vacuum		
Raman et al. [11]	7 alternating layers of 230 nm thick SiO ₂ , 485 nm thick HfO ₂ , 688 nm thick SiO ₂ , 13 nm thick HfO ₂ , 73 nm thick SiO ₂ , 34 nm thick HfO ₂ , and 54 nm thick SiO ₂ with 200 nm thick Ag solar reflector and supported by 750 μm Si layer at the back	Numerical and Field Investigation	0.970	0.65	Stanford, California, USA	4.9	40.1
					Non-vacuum		
Zhai et al. [33]	Micrometer-sized SiO ₂ spheres randomly distributed in a matrix material of polymethylpentene (TPX)	Numerical and Field Investigation	0.969	0.93	Cave creek, Arizona, USA	N.A.	93
					Non-vacuum		

This work	3 layers of 8 μm thick triangular prism patterned PDMS, 500 μm thick SiO_2 , and 120 nm thick Ag solar reflector at the back	Numerical and Field Investigation	0.975	0.98	Hong Kong, China	6.2	19.7
					Non-vacuum		

480

481 **Conflict of Interest**

482 The authors declare that there is no conflict of interest.

483

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490

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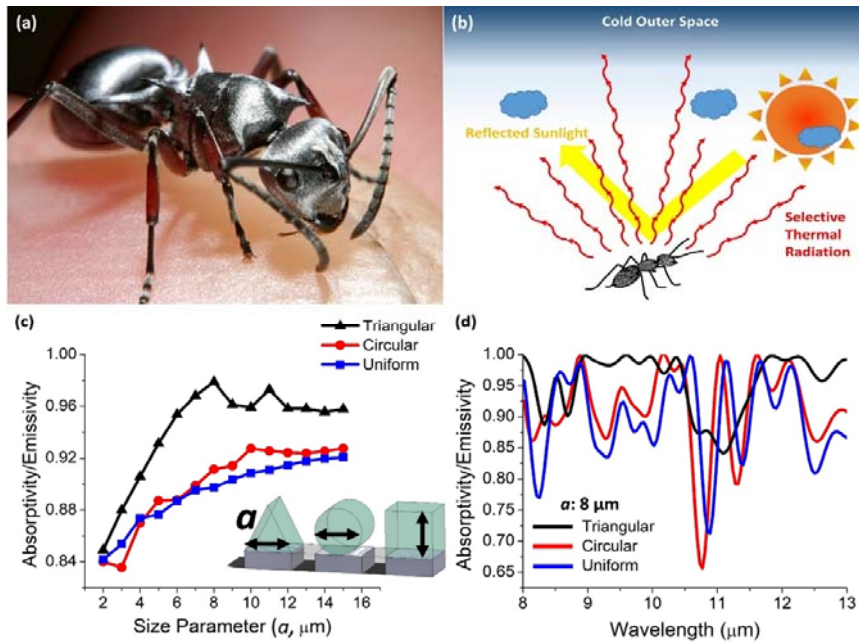
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1 Figure Captions



2

3 Fig. 1. (a) Photograph of Saharan silver ant, *Cataglyphis bombycina* (b) A schematic diagram

4 of thermoregulatory effect discovered in the Saharan silver ant (c) FDTD simulation

5 result for averaged MIR emissivity (8-13 μm) of three different geometrical

6 configurations (triangular prism arrays (black solid line), circular rod arrays (red solid

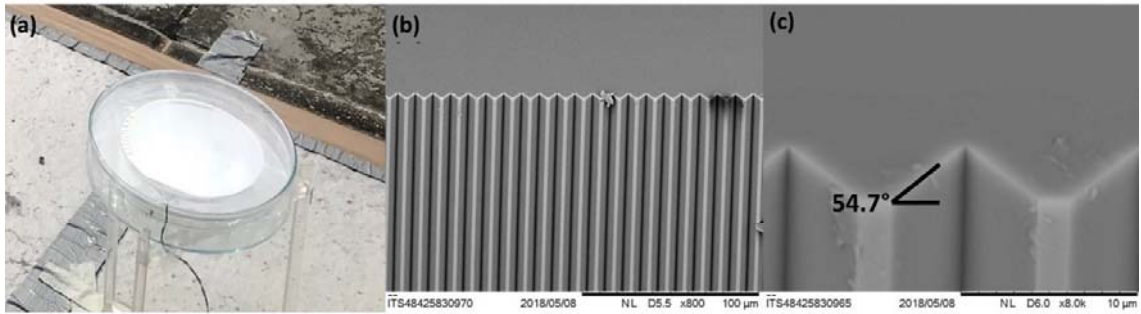
7 line), and uniform flat layer (blue solid line)) along with characteristic length of 2-15

8 μm (d) FDTD simulation result for MIR emission spectrum (8-13 μm) of emitters with

9 triangular prism arrays (black solid line), circular rod arrays (red solid line), and

10 uniform flat layer (blue solid line) at a constant characteristic length of 8 μm

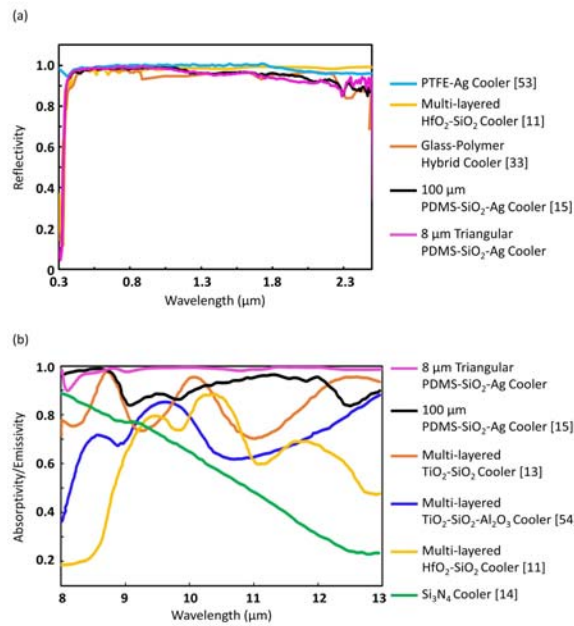
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13

14 Fig. 2. (a) A fabricated 4-inch wafer size 8 μm characteristic length triangular PDMS-SiO₂-Ag
 15 daytime passive radiative cooler under direct sunlight. (b) SEM image of the fabricated
 16 silicon mold with 8 μm characteristic length triangular arrays with a scale bar at 100
 17 μm. (c) SEM image of the fabricated silicon mold with 8 μm characteristic length
 18 triangular arrays with a scale bar at 10 μm.

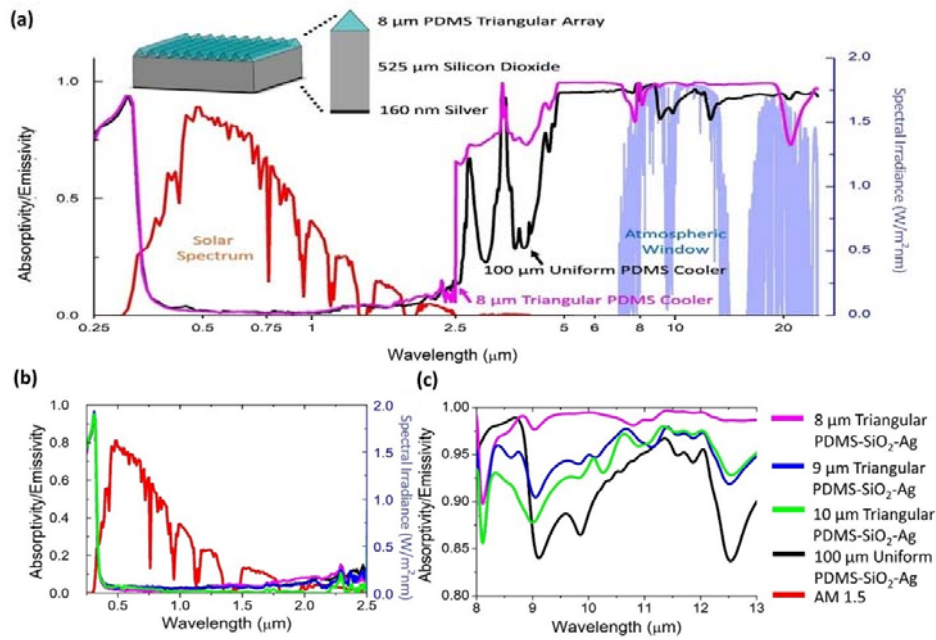
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21

22 Fig. 3. (a) Measured full emission spectrum of the uniform 100 μm thick PDMS-SiO₂-Ag
 23 (black solid line) and the patterned PDMS-SiO₂-Ag with triangular prism array of a
 24 size parameter at 8 μm (pink solid line) in the wavelength range of 0.25-25 μm. (b)
 25 Measured emissivity of the uniform 100 μm thick PDMS-SiO₂-Ag (black solid line),
 26 the patterned PDMS-SiO₂-Ag with triangular prism array of a size parameter at 8 μm
 27 (pink solid line), a size parameter at 9 μm (blue solid line), a size parameter at 10 μm
 28 (green solid line) in the wavelength range of 0.25-2.5 μm. The AM 1.5 solar spectrum
 29 (red solid line). (c) Measured emissivity of the uniform 100 μm thick PDMS-SiO₂-Ag
 30 (black solid line), the patterned PDMS-SiO₂-Ag with triangular prism arrays of a size
 31 parameter at 8 μm (pink solid line), a size parameter at 9 μm (blue solid line), a size
 32 parameter at 10 μm (green solid line) in the wavelength range of 8-13 μm.

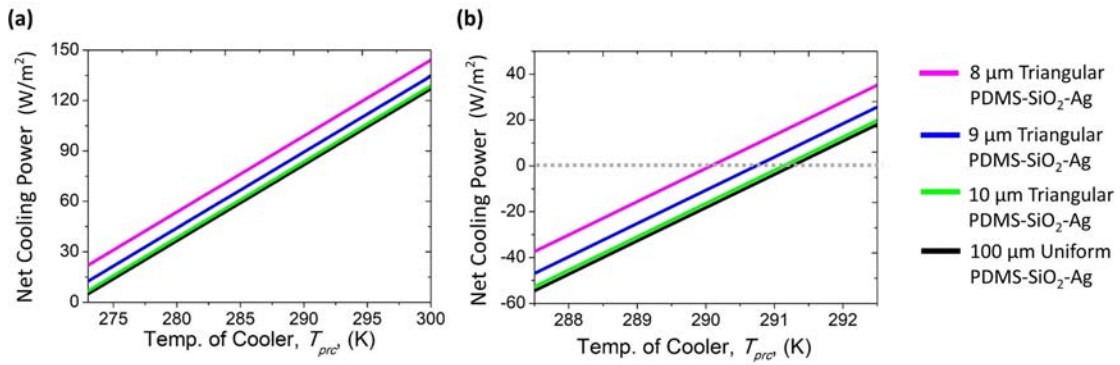
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36 Fig. 4. (a) Reflectivity spectrum within 0.3-2.5 μm of PTFE-Ag cooler [53] (light blue solid
 37 line), HfO₂-SiO₂ alternating multi-layered radiative cooler [11] (yellow solid line),
 38 glass-polymer hybrid cooler [33] (orange solid line), 100 μm uniform PDMS polymer
 39 based radiative cooler [15] (black solid line), and 8 μm triangular PDMS polymer based
 40 radiative cooler (pink solid line, this work) (b) Emission spectrum within 8-13 μm of 8
 41 μm triangular PDMS polymer based radiative cooler (pink solid line, this work), 100
 42 μm uniform PDMS polymer based radiative cooler [15] (black solid line), TiO₂-SiO₂
 43 alternating multi-layered radiative cooler [13] (orange solid line), TiO₂-SiO₂-Al₂O₃
 44 alternating multi-layered radiative cooler [54] (blue solid line), HfO₂-SiO₂ alternating
 45 multi-layered radiative cooler [11] (yellow solid line), and Si₃N₄ based radiative cooler
 46 [14] (green solid line).

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49

50 Fig. 5. (a) Numerically estimated cooling power of the uniform 100 μm thick PDMS-SiO₂-Ag

51 (black solid line), the patterned PDMS-SiO₂-Ag with triangular prism arrays of a size
52 parameter at 8 μm (pink solid line), a size parameter at 9 μm (blue solid line), a size

53 parameter at 10 μm (green solid line) with changing cooler temperature at a heat gain

54 condition of $h_{con} = 0 \text{ W}/(\text{m}^2\text{K})$. (b) Numerically estimated cooling power of the

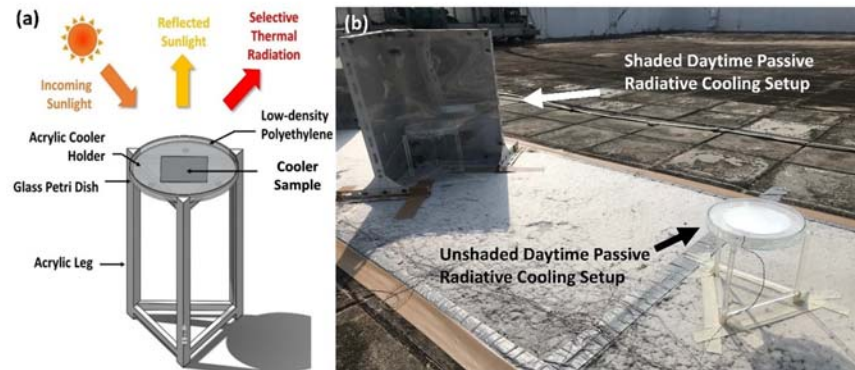
55 uniform 100 μm thick PDMS-silica-mirror (black solid line), the patterned PDMS-

56 SiO₂-Ag with triangular prism arrays of a size parameter at 8 μm (pink solid line), a

57 size parameter at 9 μm (blue solid line), a size parameter at 10 μm (green solid line)

58 with changing cooler temperature at a heat gain condition of $h_{con} = 10 \text{ W}/(\text{m}^2\text{K})$.

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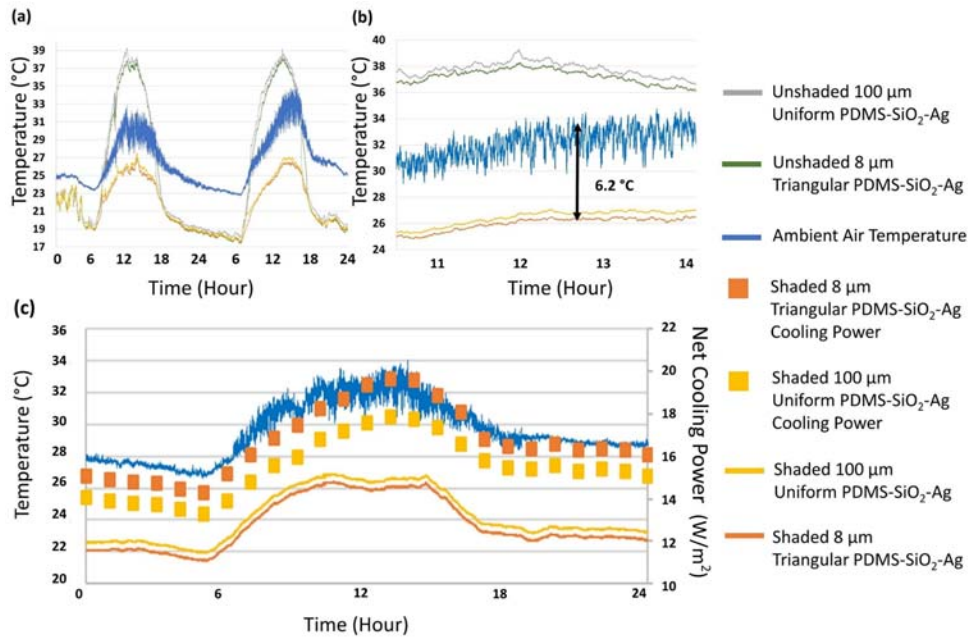
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62 Fig. 6. (a) A schematic diagram of experimental setup for daytime passive radiative cooling (b)

63 Experimental setups for both shaded and unshaded daytime passive radiative cooling.

64



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67 Fig. 7. (a) Temperature profile of the unshaded uniform 100 μm thick PDMS-SiO₂-Ag (grey

68 solid line), the unshaded 8 μm triangle PDMS-SiO₂-Ag (green solid line), and the

69 temperature of ambient (blue solid line), the shaded uniform 100 μm thick PDMS-SiO₂-

70 Ag (yellow solid line), the shaded 8 μm PDMS-SiO₂-Ag (orange), through a 48 hrs

71 cycle (Date: 30-Sep-2018 to 31-Sep-2018). The average daytime relative humidity was

72 87 % and 0-1 oktas sky condition was observed. The average global solar intensity was

73 measured to be 1010 W/m² at peak daytime (11:00 am to 13:00 pm). (b) Zoom-in of the

74 temperature profile (Date: 31-Sep-2018) with the device under direct solar irradiation.

75 The shaded 8 μm triangle PDMS-SiO₂-Ag cooler (orange solid line) achieved a

76 temperature 6.2 °C below the temperature of ambient. (c) Net cooling power (square

77 box) and temperature (line) profiles of the shaded 100 μm uniform PDMS-SiO₂-Ag

78 cooler (yellow) and the shaded 8 μm triangular patterned PDMS-SiO₂-Ag cooler

79 (orange) at the ambient air temperature (blue) during a 24-hr cycle (Date: 08-June-

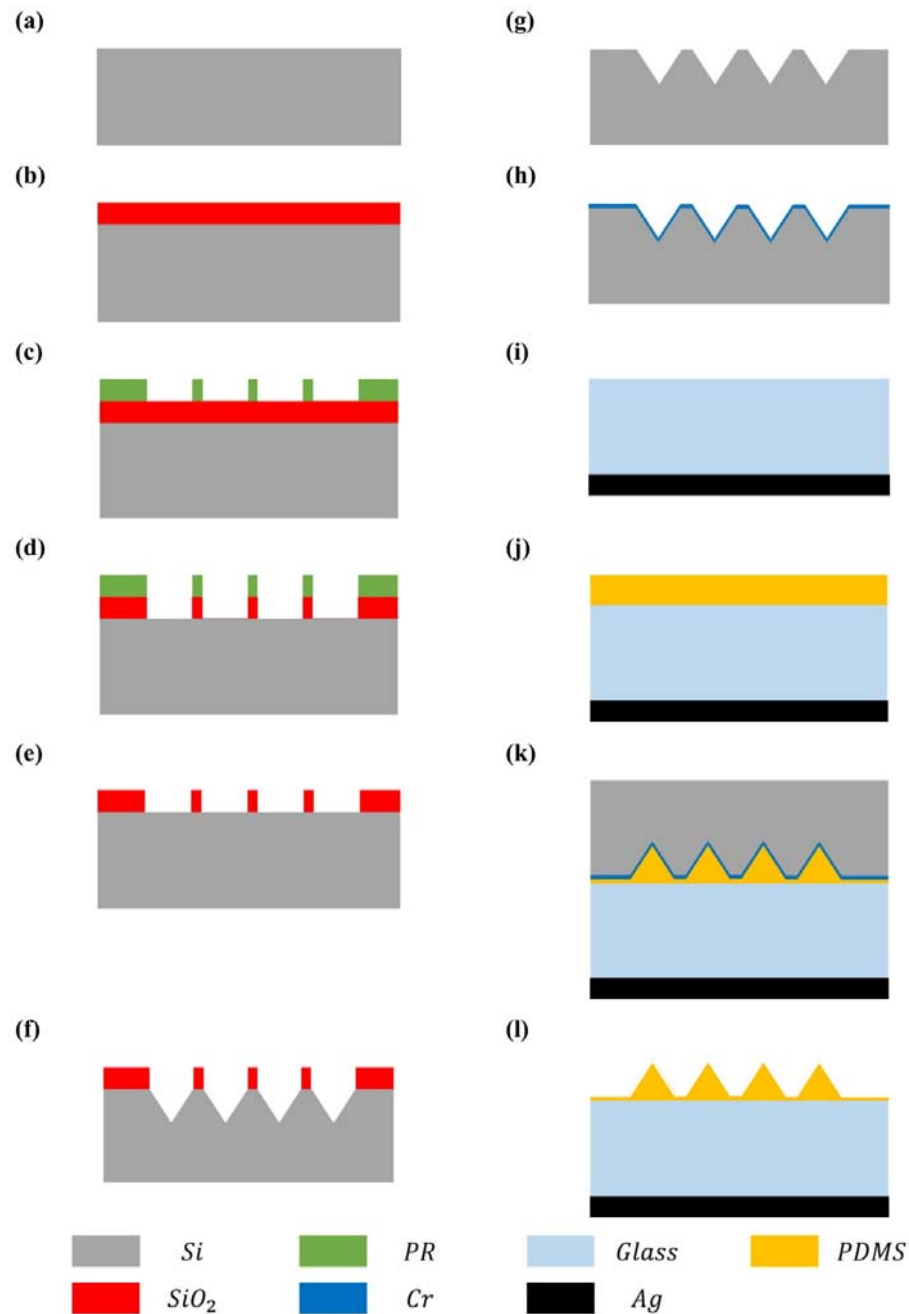
80 2019). The average relative humidity during the daytime and nighttime was 85 % and

81 65 %, respectively. 0-1 oktas sky condition was observed. The average global solar

82 intensity was measured at 1020 W/m^2 during the peak daytime (from 12:00 pm to 2:00
83 pm).

84

1 Supplementary Materials



2

3 Fig. S1. Fabrication process flow of the bio-inspired passive radiative cooler. (a) Preparation
 4 of a 525 μm thick P-type 4-inch silicon wafer cleaned by Piranha and hydrofluoric acid
 5 solution. (b) Thermal oxidation of the silicon wafer for 60 nm. (c) Photo-lithography of the
 6 pattern on the silicon wafer. (d) AOE process to etch the thermal oxide layer for 30 s. (e) Photo
 7 resist layer strip off process. (f) Silicon wet etching process with TMAH solution (g) BOE

- 8 process to remove the remaining thermal oxide layer. (h) Sputter 160 nm thickness chromium
- 9 on the surface of the mold. (i) Sputter 160 nm Silver at the back of the silicon dioxide substrate.
- 10 (j) Spin-coat 20 μm PDMS on the silicon dioxide substrate. (k) Nano-imprint process (l)
- 11 Removal of the mold after baking.
- 12